

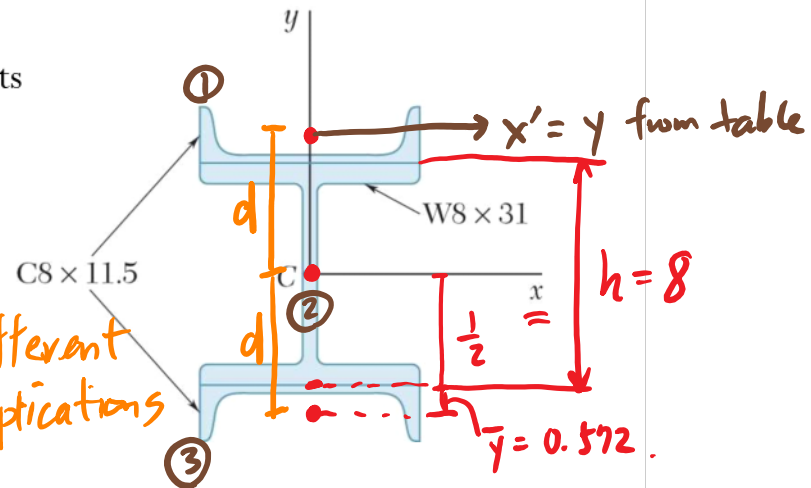
Announcements

- CBTF Quiz 5 continues.

□ Upcoming deadlines:

- Friday (4/19): Written Assignment
- Tuesday (4/23): PL HW



$$\begin{aligned} I_x &= I_{x'} + md^2 \\ &\quad + Ad^2 \\ &\quad + Vd^2 \end{aligned}$$


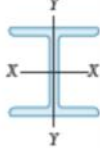
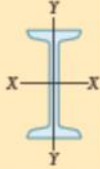
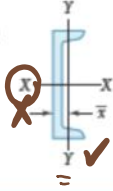
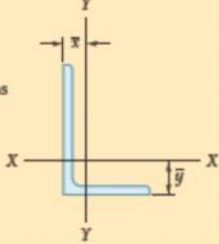
$$= (I_{x1'} + A_1 d^2) + I_{x2'} + (I_{x3'} + A_3 d^2)$$

$$I_{x1'} = I_{x3'} = 1.31 \text{ in}^4$$

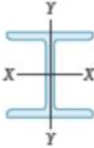
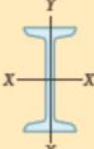
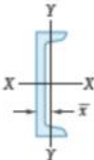
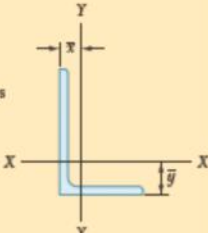
$$I_{xz'} = 110 \text{ in}^4$$

$$d = \frac{1}{2}(8 \text{ in}) + 0.572 \text{ in} = 4.572 \text{ in}$$

$$A_1 = A_3 = 3.37 \text{ in}^2$$

	Designation	Area in ²	Depth in.	Width in.	Axis X-X			Axis Y-Y		
					\bar{I}_x , in ⁴	\bar{k}_x , in.	\bar{y} , in.	\bar{I}_y , in ⁴	\bar{k}_y , in.	\bar{x} , in.
W Shapes (Wide-Flange Shapes) 	W18 × 76†	22.3	18.2	11.0	1330	7.73		152	2.61	
	W16 × 57	16.8	16.4	7.12	788	6.72		43.1	1.60	
	W14 × 38	11.2	14.1	6.77	385	5.87		26.7	1.55	
	W8 × 31	9.12	8.00	8.00	110	3.47		37.1	2.02	
S Shapes (American Standard Shapes) 	S18 × 54.7†	16.0	18.0	6.00	801	7.07		20.7	1.14	
	S12 × 31.8	9.31	12.0	5.00	217	4.83		9.33	1.00	
	S10 × 25.4	7.45	10.0	4.66	123	4.07		6.73	0.950	
	S6 × 12.5	3.66	6.00	3.33	22.0	2.45		1.80	0.702	
C Shapes (American Standard Channels) 	C12 × 20.7†	6.08	12.0	2.94	129	4.61		3.86	0.797	0.698
	C10 × 15.3	4.48	10.0	2.60	67.3	3.87		2.27	0.711	0.634
	C8 × 11.5	3.37	8.00	2.26	32.5	3.11		1.31	0.623	0.572
Angles 	L6 × 6 × 1†	11.0			35.4	1.79	1.86	35.4	1.79	1.86
	L4 × 4 × 1/2	3.75			5.52	1.21	1.18	5.52	1.21	1.18
	L3 × 3 × 1/4	1.44			1.23	0.926	0.836	1.23	0.926	0.836
	L6 × 4 × 1/2	4.75			17.3	1.91	1.98	6.22	1.14	0.981
	L5 × 3 × 1/2	3.75			9.43	1.58	1.74	2.55	0.824	0.746
	L3 × 2 × 1/4	1.19			1.09	0.933	0.980	0.390	0.569	0.487

$A d^2$
 $[\text{in}^2][\text{in}]^2$
 md^2
 $[\text{kg}][\text{m}]^2$

		Designation	Area mm ²	Depth mm	Width mm	Axis X-X			Axis Y-Y		
						\bar{I}_x 10 ⁶ mm ⁴	\bar{k}_x mm	\bar{y} mm	\bar{I}_y 10 ⁶ mm ⁴	\bar{k}_y mm	\bar{x} mm
W Shapes (Wide-Flange Shapes)		W460 × 113†	14400	462	279	554	196		63.3	66.3	
		W410 × 85	10900	417	181	316	171		17.9	40.6	
		W360 × 57.8	7230	358	172	160	149		11.1	39.4	
		W200 × 46.1	5890	203	203	45.8	88.1		15.4	51.3	
S Shapes (American Standard Shapes)		S460 × 61.4†	10300	457	152	333	180		8.62	29.0	
		S310 × 47.3	6010	305	127	90.3	123		3.88	25.4	
		S250 × 37.8	4810	254	118	51.2	103		2.80	24.1	
		S150 × 18.6	2360	152	84.6	9.16	62.2		0.749	17.8	
C Shapes (American Standard Channels)		C310 × 30.8†	3920	305	74.7	53.7	117		1.61	20.2	17.7
		C250 × 22.8	2990	254	66.0	25.0	98.3		0.945	18.1	16.1
		C200 × 17.1	2170	203	57.4	13.5	79.0		0.545	15.8	14.5
		C150 × 12.2	1540	152	48.8	5.45	59.4		0.296	13.6	13.0
Angles		L152 × 152 × 25.4†	7100			14.7	45.5	47.2	14.7	45.5	47.2
		L102 × 102 × 12.7	2420			2.30	30.7	30.0	2.30	30.7	30.0
		L76 × 76 × 6.4	929			0.512	23.5	21.2	0.512	23.5	21.2
		L152 × 102 × 12.7	3060			7.20	48.5	50.3	2.59	29.0	24.9
		L127 × 76 × 12.7	2430			3.93	40.1	44.2	1.06	20.9	18.9
		L76 × 51 × 6.4	768			0.454	24.2	24.9	0.162	14.5	12.4

Determine the moments of inertia of the bracket with respect to the x- and y-axes.

① & ② : square plates ; ③ & ④ disks.

$$I_x = I_{1x} + I_{2x} - I_{3x} - I_{4x}$$

- Since the x-axis goes through the centroid of ①, $I_{1x} = I_{1x'}$ (same for I_{3x} , but not ② & ④)

$$I_x = I_{1x'} + (I_{2x'} + A_2 d_2^2) - I_{3x'} - (I_{4x'} + A_4 d_4^2)$$

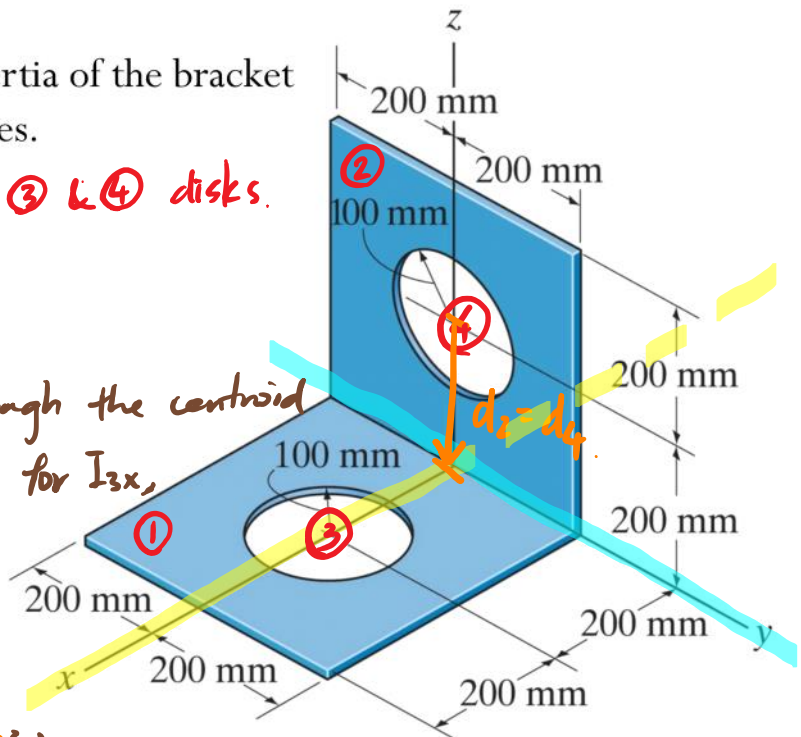


Figure: 10_P106-107

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$$I_{1x'} = \frac{1}{12} m_1 (400)^2$$

$$I_{2x'} = \frac{1}{12} m_2 (400^2 + 400^2)$$

$$I_{3x'} = \frac{1}{4} m_3 (100)^2$$

$$I_{4x'} = \frac{1}{2} m_4 (100)^2$$

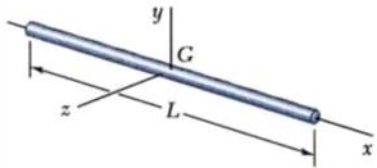
$$A_2 = (400)^2 \text{ mm}^2$$

$$A_4 = \pi (100)^2 \text{ mm}^2$$

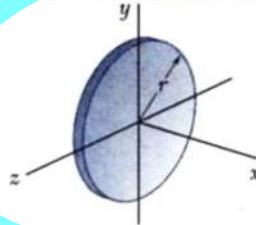
$$d_2 = d_4 = 200 \text{ mm}$$

Vector Mechanics for Engineers: Statics

Moments of Inertia of Common Geometric Shapes

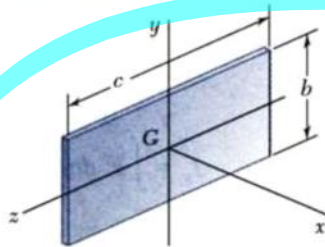


$$I_y = I_z = \frac{1}{12} mL^2$$



$$I_x = \frac{1}{2} mr^2$$

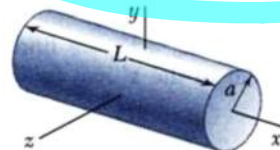
$$I_y = I_z = \frac{1}{4} mr^2$$



$$I_x = \frac{1}{12} m(b^2 + c^2)$$

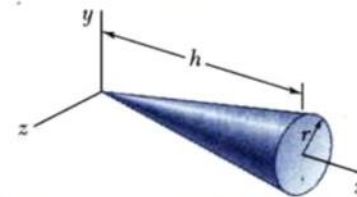
$$I_y = \frac{1}{12} mc^2$$

$$I_z = \frac{1}{12} mb^2$$



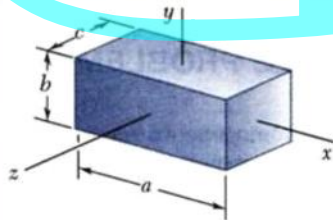
$$I_x = \frac{1}{2} ma^2$$

$$I_y = I_z = \frac{1}{12} m(3a^2 + L^2)$$



$$I_x = \frac{3}{10} ma^2$$

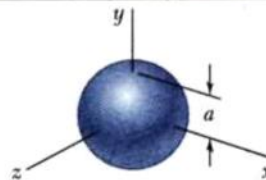
$$I_y = I_z = \frac{3}{5} m(\frac{1}{4} a^2 + h^2)$$



$$I_x = \frac{1}{12} m(b^2 + c^2)$$

$$I_y = \frac{1}{12} m(c^2 + a^2)$$

$$I_z = \frac{1}{12} m(a^2 + b^2)$$



$$I_x = I_y = I_z = \frac{2}{5} ma^2$$

Virtual Work

Main goals and learning objectives

- Introduce the principle of virtual work
- Show how it applies to determining the equilibrium configuration of a series of pin-connected members

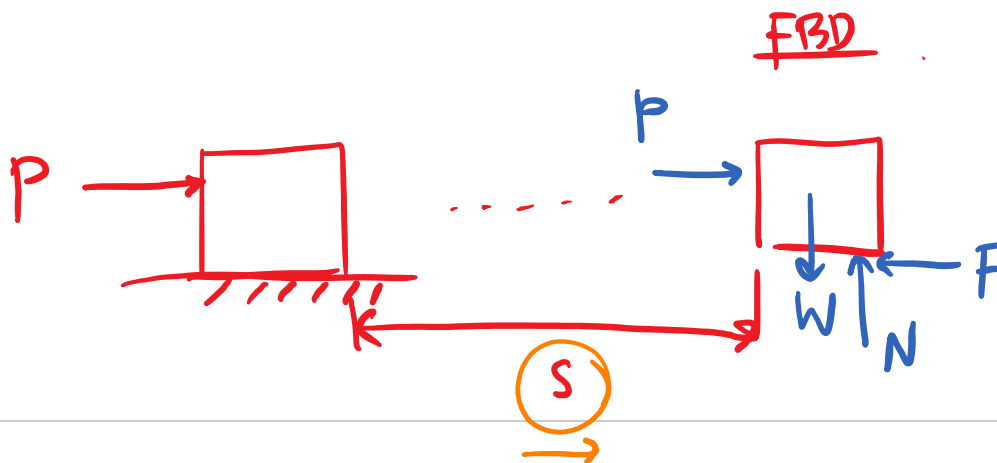
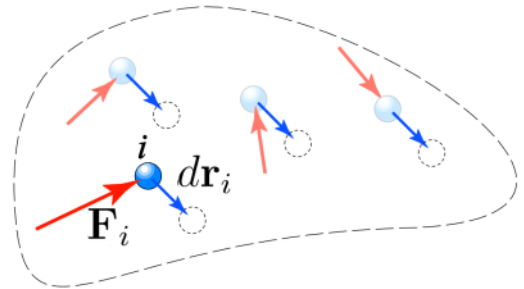
Definition of Work

Work of a force

A force does work when it undergoes a displacement in the direction of the line of action.

The work dU produced by the force \mathbf{F} when it undergoes a differential displacement $d\mathbf{r}$ is given by

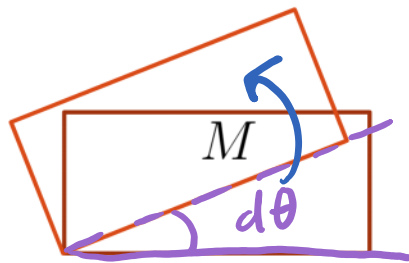
$$dU = \mathbf{F} \cdot d\mathbf{r}$$



W & N : did no work.
P & F : did work.

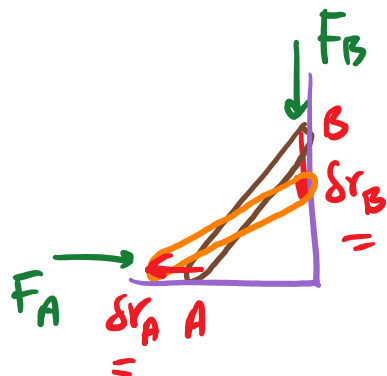
Definition of Work

Work of a couple $dU = M \mathbf{k} \cdot d\theta \mathbf{k} = \underline{\underline{M d\theta}}$



Virtual Displacements

A *virtual displacement* is a conceptually possible displacement or rotation of all or part of a system of particles. The movement is assumed to be possible, but actually does not exist.

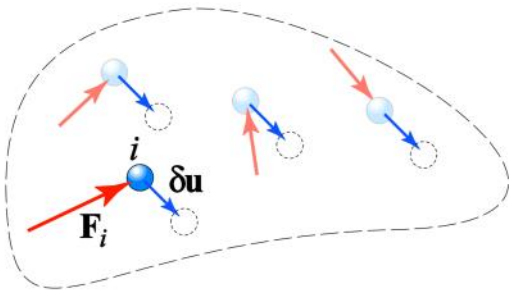


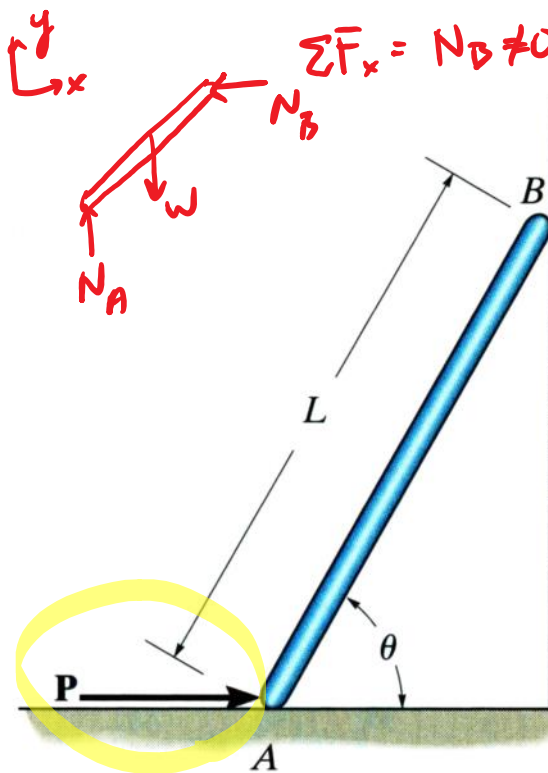
Principle of Virtual Work

The principle of virtual work states that if a body is in equilibrium, then the algebraic sum of the virtual work done by all the forces and couple moments acting on the body is zero for any virtual displacement of the body. Thus,

$$\sum \mathbf{F}_i \cdot d\mathbf{u}_i + \sum M_i \cdot d\theta_i = 0$$

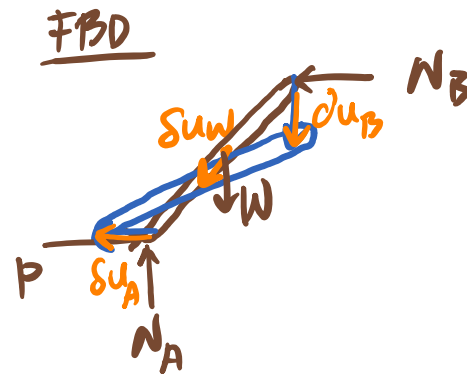
for a body at equilibrium.





$\Sigma \vec{F}_x = N_B \neq 0 \rightarrow P$ is required for equilibrium.

The thin rod of weight W rests against the smooth wall and floor. Determine the magnitude of force P needed to hold it in equilibrium.



Virtual work eqn.

$$\Sigma U_i = 0 = U_P + U_W$$

$$U_P = \vec{P} \cdot \delta \vec{u}_A$$

$$U_W = \vec{W} \cdot \delta \vec{u}_G \rightarrow W_u \delta u_G$$

$N_A: \perp \delta u_A \rightarrow$ no work.

$N_B: \perp \delta u_B \rightarrow$ no work.

$W: \text{do some work.}$

$P: \parallel \delta u_A \rightarrow$ do work.

