Statics - TAM 210 & TAM 211

Lecture 21
March 5, 2018
Chap 7.2

Announcements

- ☐ Upcoming deadlines:
- Monday (3/5)
 - Mastering Engineering Tutorial 8
- Tuesday (3/6)
 - PL HW 6
- Quiz 4 (3/7-9)
 - Sign up at CBTF
 - Up thru and including Lecture 19 (Frames & Machines). Note that quiz and lecture material always builds on earlier fundamental concepts.
- Quiz 5 (3/14-16)
- No class Friday March 9, enjoy EOH!
- No Prof. H-W office hours on Friday March 9
- If building is picketed, no class Wednesday March 7. Will have online video

Chapter 7: Internal Forces

Goals and Objectives

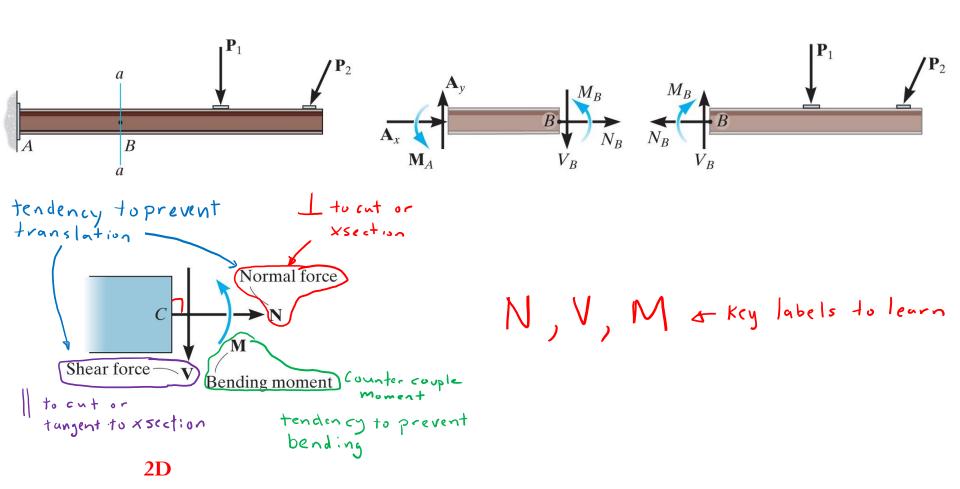
- Determine the internal loadings in members using the method of sections
- Generalize this procedure and formulate equations that describe the internal shear and bending moment throughout a member
- Be able to construct or identify shear and bending moment diagrams for beams when distributed loads, concentrated forces, and/or concentrated couple moments are applied

Recap: Internal loadings in structural members

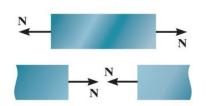
Structural Design: need to know the loading acting within the member in order to be sure the material can resist this loading

Cutting members at internal points reveal internal forces and moments.

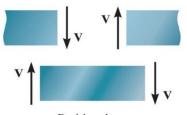
We Method of Sections



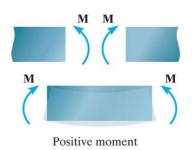
Recap: Sign conventions:



Positive normal force

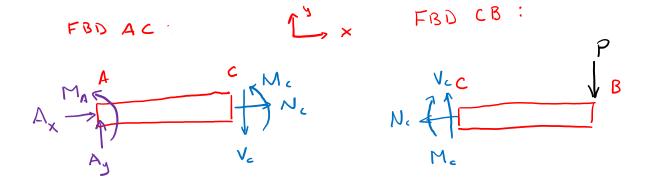


Positive shear



A fixed (contilever) support

If beam AB is cut at C, draw FBDs of sections AC, CB illustrating assumptions of N, V, M drawn in positive directions.

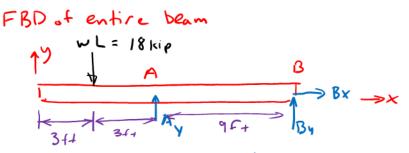


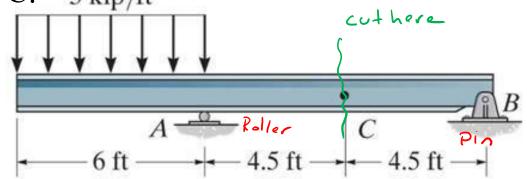
Note: a Ithough draw Voffthe side of the cut section, V is actually applied at the cut.

Recap: Procedure for analysis:

- 1. Find support reactions (free-body diagram of entire structure)
- 2. Pass an imaginary section through the member
- 3. Draw a free-body diagram of the segment that has the least number of loads on it
- 4. Apply the equations of equilibrium

Find the internal forces at point C. 3 kip/ft

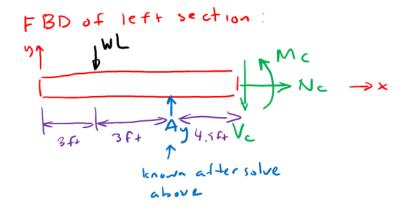




3 unknowns (Ay, Bx, By)

use 3 Eo = to solve for Ay, Bx, By.

$$\Sigma F_{x} \cdot B_{x} = 0$$
, $\Sigma F_{y} \cdot A_{y} + B_{y} - WL = 0$
+ $\int ZM_{g} \cdot (12ft) UL - (9ft) A_{y} = 0 \rightarrow A_{y} = 24 k \cdot r$

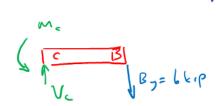


3 unknowns (
$$N_c, V_c, M_c$$
), assuming knowny use $E \circ E$:

 $EF_x : N_c = 0$
 $EF_y : N_c = 0$
 $EF_y : A_y - wL - V_c = 0 \Rightarrow V_c = 6 \text{ kip}$
 $EF_y : M_c - (4 \text{ s.f.}) A_y + (7 \text{ s.f.}) wL = 0$
 $EF_z : M_c = -27 \text{ kip.f.}$

Find the internal forces at point C. 3 kip/ft cut here FBD of entire beam 3 unknowns (Ay, Bx, By) use 3 Eo E to solve for Ay, Bx, By Ay = 24 kip Bx = 0 By = -6 kip Alternatively, could examine right section: USE EOE: FBD of right section EFx : Bx -Nc = 0 ⇒ [Nc=0] N_{-} $\xrightarrow{\uparrow_{N_{-}}}$ g_{x} $\xrightarrow{\uparrow_{N_{-}}}$ χ ZFy: By + Vc = O > Vc = 6 kip + 1 2 M2: - M2 + (4.5H) By = 0 Bunkawas (Nc, Vc, Mc) assuming → Mc = -27 kip.++ Know Bx ,By

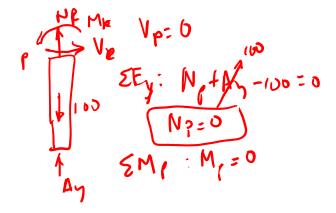
: Actual Forces & Moments:



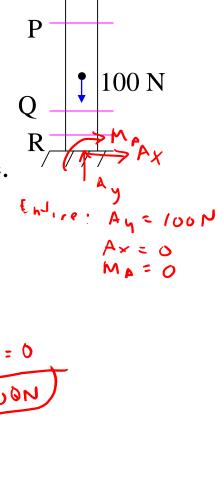
Note changes in directions of arrows for By & Mc from original FBDs due to negative values in solutions.

- 1. A column is loaded with a vertical 100 N force. At which sections are the internal loads the same?
 - A) P, Q, and R





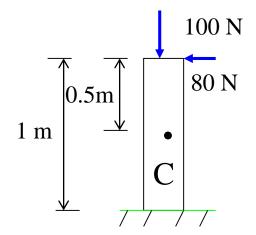
- B) P and Q
- D) None of the above.



2. Determine the magnitude of the internal loads (normal, shear, and bending moment) at point C.

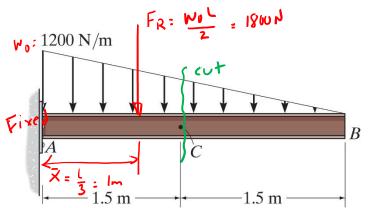
NVM

- A) (100 N, 80 N, 80 N m)
- B) (100 N, 80 N, 40 N m)
- C) (80 N, 100 N, 40 N m)
- D) (80 N, 100 N, 0 N m)

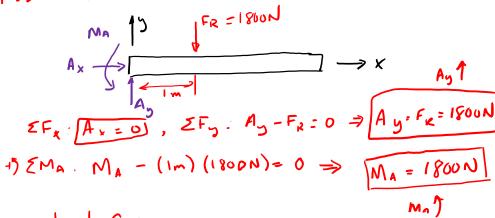




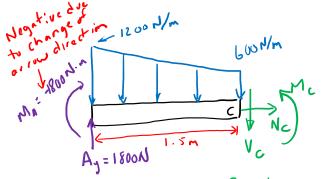
Find the internal forces and moments at C



FBD of entire bean.



Let's look at FBDs of Lett & Right sides when cut at C:

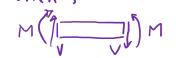


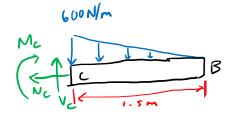
3unknowns

force & noment arrows Nc, Vc, Mc

following positive

shear & bending





3 unknowns Nc, Vc, Mc

We can solve for unknown internal forces with either left or right side: Left side: Right side: Divide distributed load into simply find Fe for distributed FRI for triangle and FRZ for rectangle Wa1 : (00N/m 600 N/m Wol : 600 N (1.5m) = 450 N A = 1800N $\Sigma F_{x} : -N_{c} = 0 \Rightarrow N_{c} = 0$ FR = Wo, L = 480 N MA= -1800 Nm (4) -EFy: Vc - 450N → Vc = 456N 47 EMc: -Mc - (0.5~)(450W) = 0 Mc = -225 N m) Mc (47= 1800Y Note that choosing left FBD EFx: [Nc = 0] takes more steps, but get the same result. EF5: Ay - FR, -FR, -Vc = 0 What are internal Nº = 1800N - A20N - JOON forces along the length Vc = 450N \ V.1 +9 EMA: -MA - (0,5 m) FRI - (0.75n) FRZ - (1.5n) Vc +Mc = 0 of the beam? Mc = -225 Nm Since negative = assumed across direction on FBD is incorrect; should be & Mc

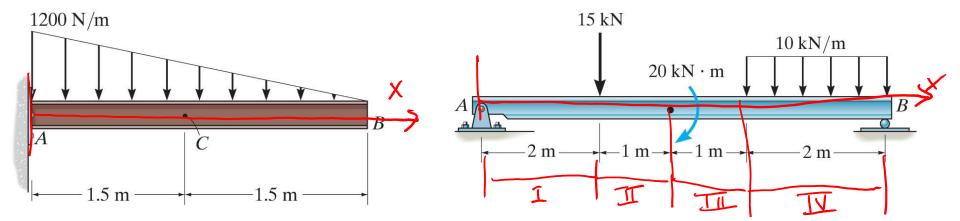
Shear Force and Bending Moment Diagrams

Goal: provide detailed knowledge of the variations of internal shear force and bending moments (V and M) throughout a beam when perpendicular distributed loads, concentrated forces, and/or concentrated couple moments are applied.

Normal forces (N) in such beams are zero, so we will not consider normal force diagrams.

Procedure

- 1. Find support reactions (free-body diagram of entire structure)
- 2. Specify coordinate *x* (start from left)
- 3. Divide the beam into sections according to loadings
- 4. Draw FBD of a section
- 5. Apply equations of equilibrium to derive V and M as functions of x(V(x)) M(x)



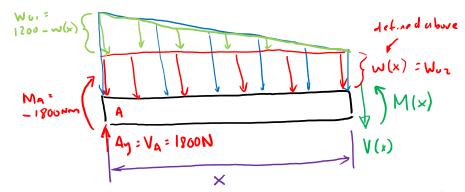
Draw the shear and bending moment diagrams for the beam. From previous example, we know that the support

From provious example, we know that the support reactions are: A=0, Ay= 1800N1, Ma=-1800Nm?

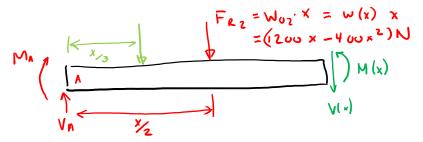
We are interested in finding V(x) & M(x) as these vary along the length of the beam.

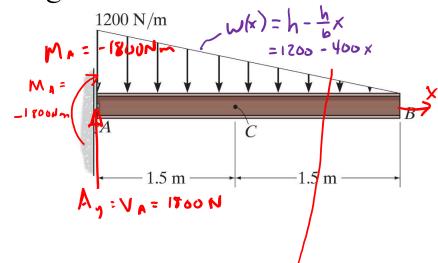
So for any length x of the beam, we get the following generic FBD as a function of x.

1200Wm



$$F_{R_1} = W_{0_1} \frac{L}{2} = [1200 - W(x)] \frac{x}{2} = (200 x^2)N$$





$$ZF_{x}: A_{y} - F_{R_{1}} - F_{R_{2}} - V(x) = 0$$

$$V(x) = (200 x^{2} - 1200 x + 1860) N$$
Guadratic

Boundary cond. trans:
$$V(x=0) = 1800 N = A_{y}$$

$$V(x=1:3m) = 0 N$$

$$cf. V(@c = 15m) = 450 N \times w/previous$$

$$evample$$

$$f ZM_{A}: -M_{A} - (\frac{x}{3})F_{R_{1}} - (\frac{x}{2})F_{R_{2}} - x \cdot V(x) + M(x) = 0$$

$$M(x) = (\frac{200}{3} x^{3} - 600 x^{2} + 1800 x - 1800) Nm$$

$$3^{rd} Order Polynomical$$

$$BC$$

$$M(v) = -1800 Nm = M_{A}$$

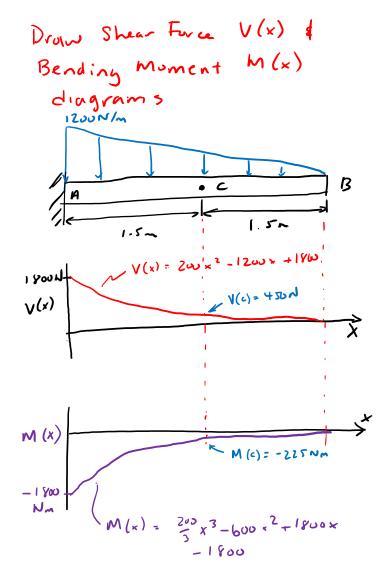
$$3^{rd} \text{ Order Polynomial}$$

$$B C$$

$$M(u) = -1860 \text{ Nm} = M_A$$

$$M(u) = 0$$

$$cf. M(Q C = 1.5m) = -225 \text{ Nm} \text{ Vm/previous}$$



Note that since the applied load is a single distributed load along the entire length of the beam, then V(x) and M(x) are continuous functions. We will see that V(x) and M(x) will be discontinuous functions when multiple loads are applied to a beam, and these discontinuities will happen at the transitions between loading regions.