



Announcements

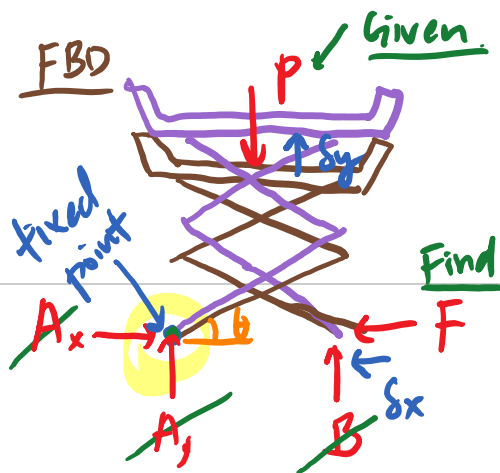
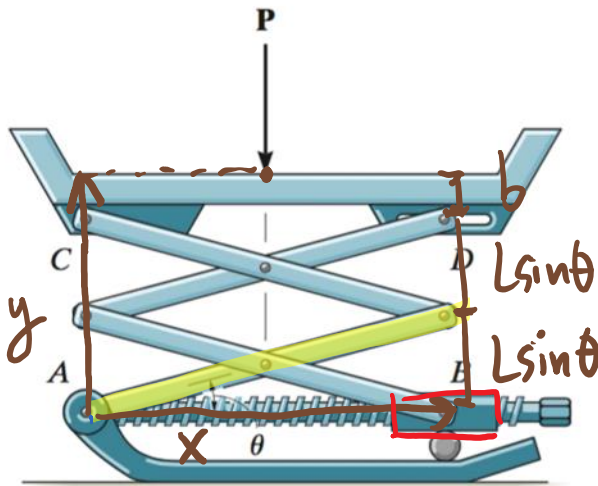
- CBTF Quiz 7 – last day!
- Last day of office hours and Piazza help: Wed, Dec. 13
- No discussion sections next week
- TAM 211 final exam starts next Thursday (12/14)

□ Upcoming deadlines:

- Saturday (12/9)
 - ME HW27
- Tuesday (12/12)
 - PL HW26



The scissors jack supports a load P . Determine the axial force in the screw necessary for equilibrium when the jack is in the position shown. Each of the four links has a length L and is pin-connected at its center. Points B and D can move horizontally.



⇒ If we use EoE to solve, analysis on individual members are needed.

Virtual Work Eq

$$\delta U = 0 = \vec{P} \cdot \delta \vec{y} + \vec{F} \cdot \delta \vec{x}$$

$$\vec{y} = 2L \sin \theta + b$$

$$\vec{P} = -P \hat{j}$$

$$\delta \vec{y} = 2L \cos \theta \delta \theta + \delta \vec{j}$$

$$\vec{F} = -F \hat{i}$$

$$\delta \vec{x} = -L \sin \theta \delta \theta \hat{i}$$

$$\vec{x} = L \cos \theta$$

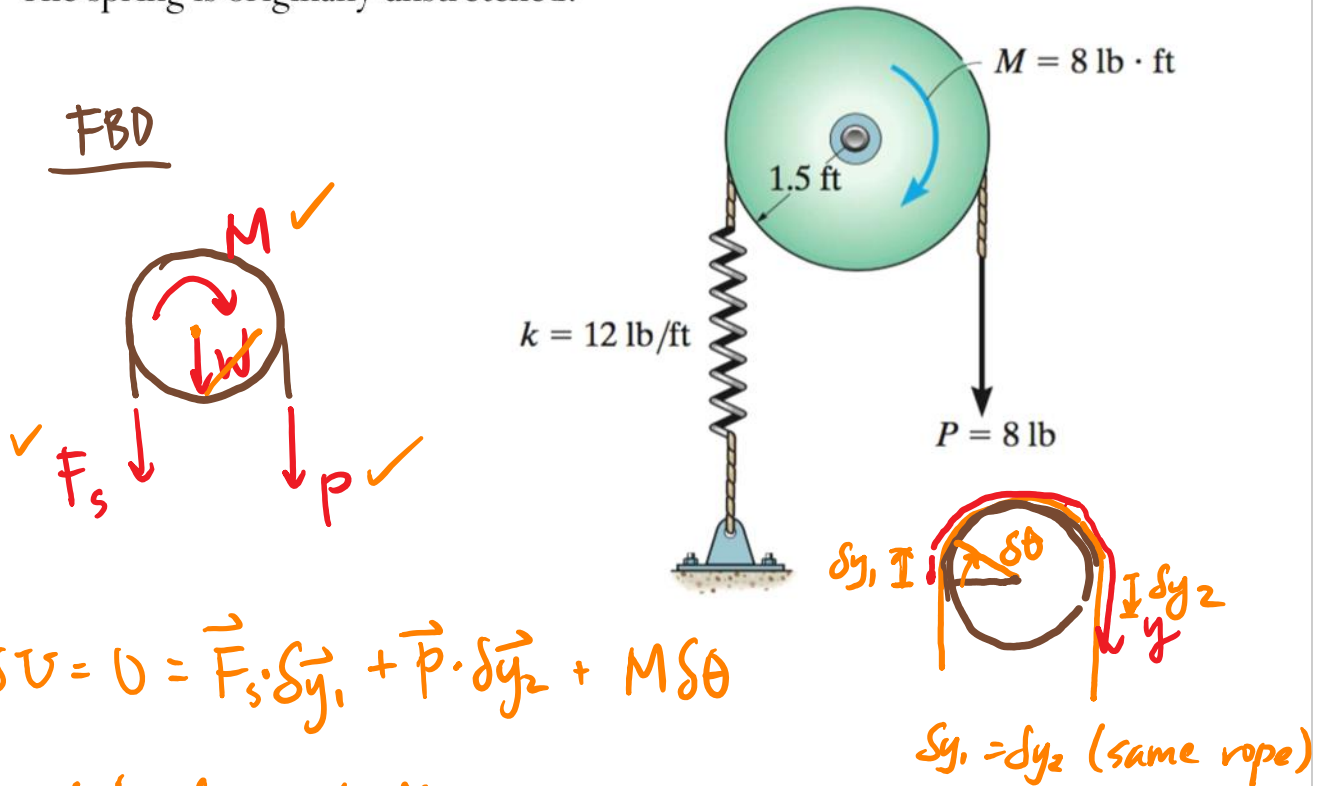
$$\delta \vec{x} = -L \sin \theta \delta \theta \hat{i}$$

$$\Rightarrow \delta U = -P(2L \cos \theta \delta \theta) + (-F)(-L \sin \theta \delta \theta) = 0$$

$$(-2PL \cos \theta + FL \sin \theta) \delta \theta = 0.$$

$$\Rightarrow \boxed{F = \frac{2P \cos \theta}{\sin \theta}}$$

The disk has a weight of 10 lb and is subjected to a vertical force $P = 8$ lb and a couple moment $M = 8$ lb ft. Determine the disk's rotation θ if the end of the spring wraps around the periphery of the disk as the disk turns. The spring is originally unstretched.



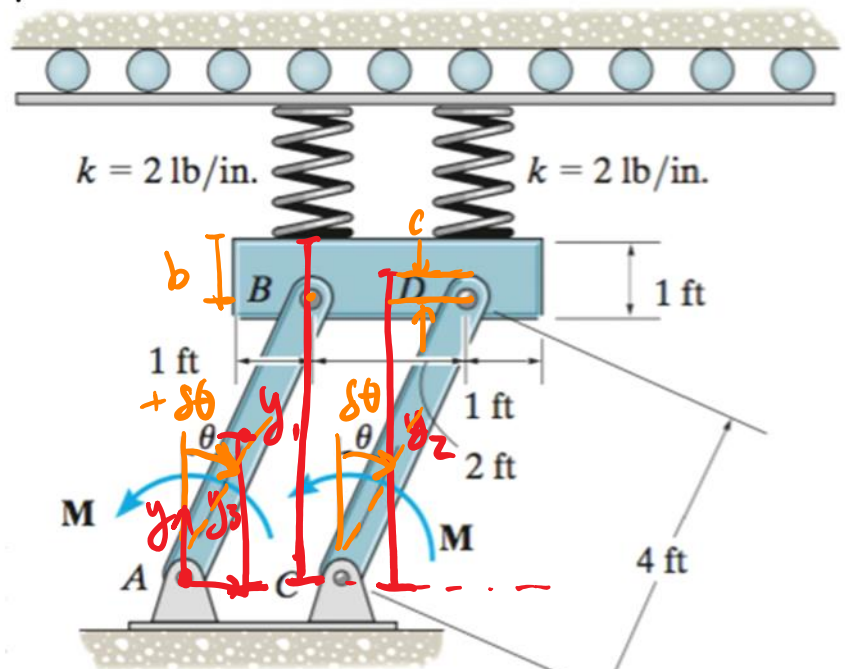
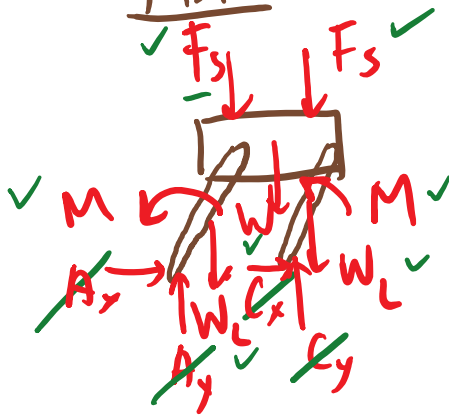
$$F_s = \frac{Pr + M}{r} = ks \quad \left. \begin{array}{l} s = y = r\theta \end{array} \right\} s = r\theta = \frac{Pr + M}{rk}$$

$$\boxed{\theta = \frac{Pr + M}{rk}} \text{ rad.}$$

When $\theta = 20^\circ$, the 50-lb uniform block compresses the two vertical springs 4 in. If the uniform links AB and CD each weigh 10 lb, determine the magnitude of the applied couple moments M needed to maintain equilibrium when $\theta = 20^\circ$.

Given: θ, k, s, W, W_L

FBD



→ Moment is doing negative work

$$y_1 = L \cos \theta + b \quad \delta y_1 = -L \sin \theta \delta \theta$$

$$y_2 = L \cos \theta + c \quad \delta y_2 = -L \sin \theta \delta \theta$$

$$y_3 = \frac{1}{2} L \cos \theta \quad \delta y_3 = -\frac{1}{2} L \sin \theta \delta \theta$$

$$\delta U = (-2M \delta \theta)$$

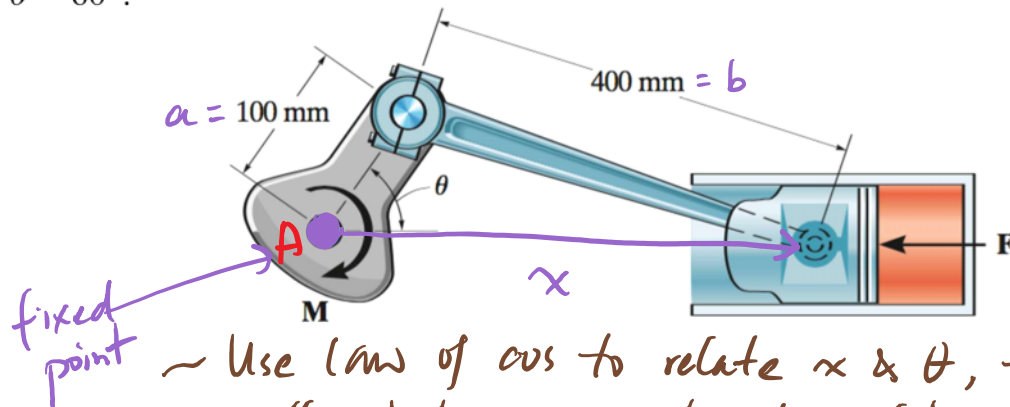
$$+ (-W)(-L \sin \theta \delta \theta)$$

$$+ 2(-F_s)(-L \sin \theta \delta \theta)$$

$$+ 2(W_L)(-\frac{1}{2} L \sin \theta \delta \theta) = 0, \quad F_s = ks$$

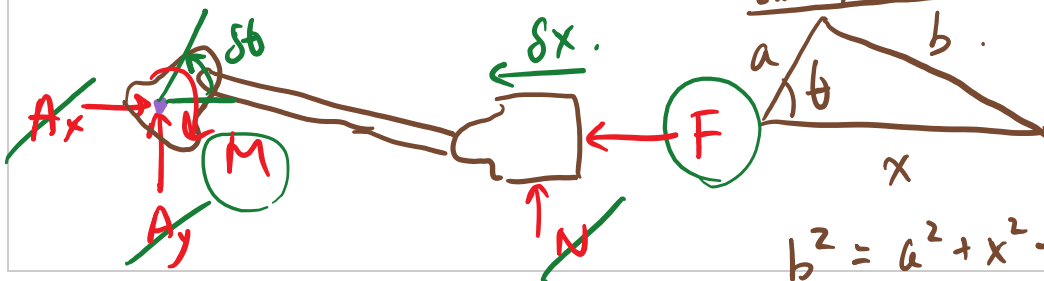
$$\Rightarrow M = \frac{1}{2} L \sin \theta (W + 2ks + W_L) = \underline{52 \text{ lb} \cdot \text{ft}}$$

The crankshaft is subjected to a torque of $M = 50 \text{ N m}$. Determine the horizontal compressive force F applied to the piston for equilibrium when $\theta = 60^\circ$.



Use law of cos to relate x & θ , then differentiate the equation to relate δx & $\delta \theta$.

FBD



Law of cos



$$b^2 = a^2 + x^2 - 2ax \cos \theta$$

• differentiate:

$$0 = 0 + 2x \delta x - 2a \cos \theta \delta x - 2ax (-\sin \theta) \delta \theta$$

Virtual Work Equation:

$$\delta U = 0 = -M \delta \theta + (-F) \delta x$$

$$= -M \delta \theta + (-F) \left(\frac{-2ax \sin \theta}{2x - 2a \cos \theta} \right) \delta \theta = 0$$

$$\delta x = \left(\frac{-2ax \sin \theta \delta \theta}{2x - 2a \cos \theta} \right)$$

$$\Rightarrow \left(-M + \frac{2Fax \sin \theta}{2x - 2a \cos \theta} \right) = 0 \Rightarrow \boxed{F = 512 \text{ N}}$$