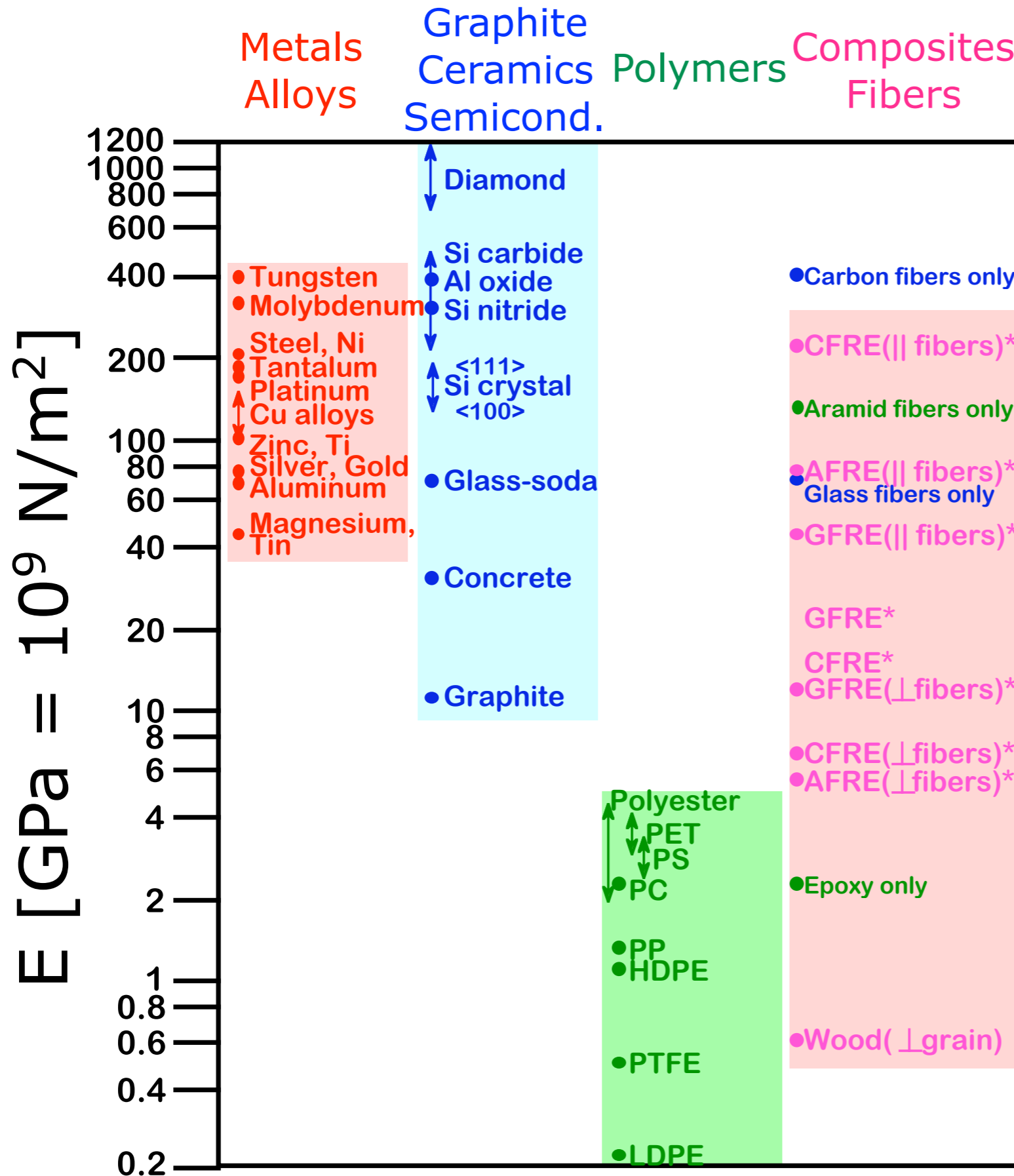


Isotropic linear elastic response

Range of Young's modulus



What is the source of both **universality** and **range** in modulus?

Based on data in Table B2, Callister 6Ed.

Composite data based on reinforced epoxy with 60 vol% of aligned carbon (CFRE), aramid (AFRE), or glass (GFRE) fibers.



Universality of linear elastic response

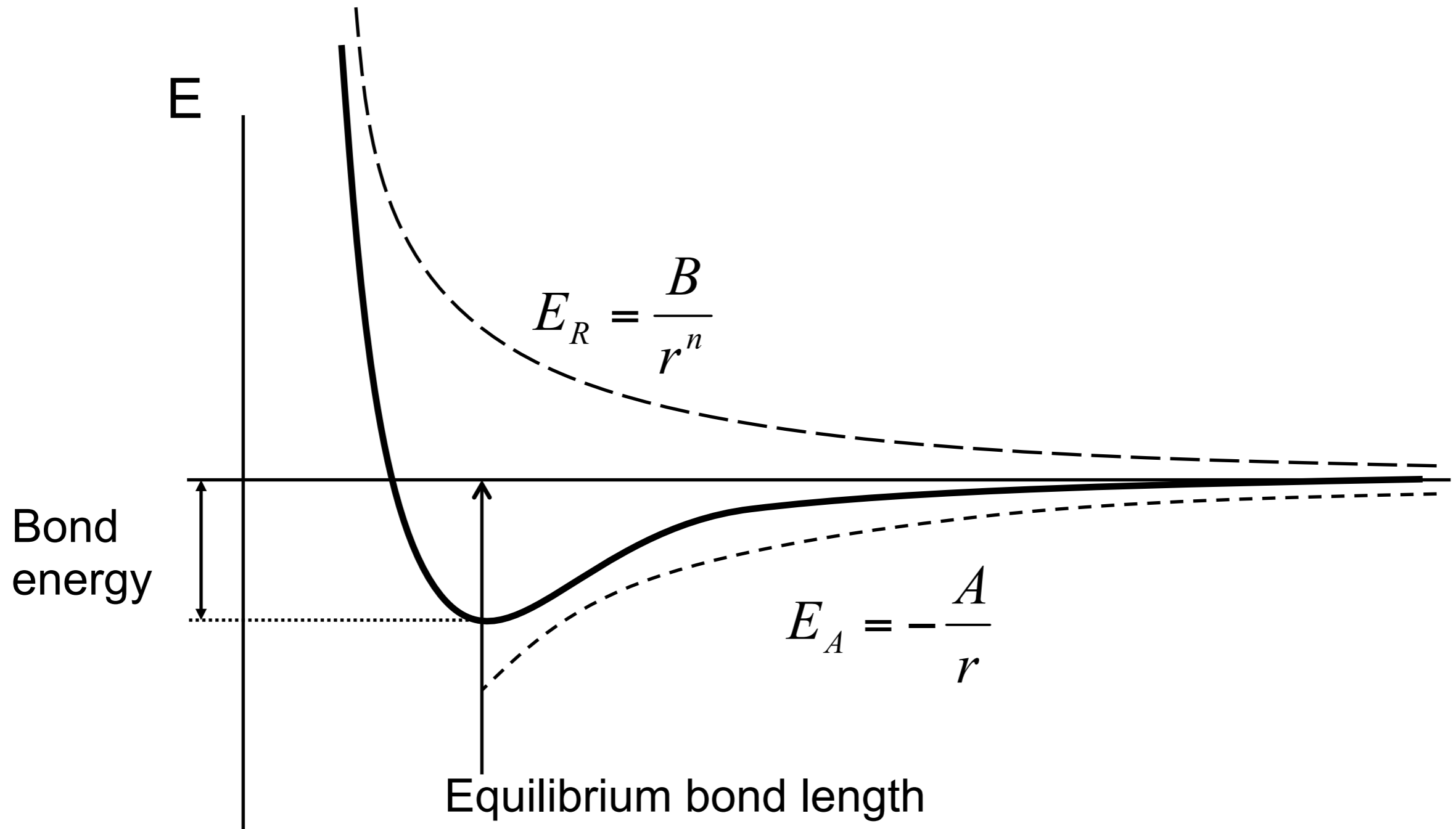
Materials are made of atoms, held together by atomic interactions

- covalent and ionic bonding: ceramics, semiconductors (~ 200 N/m)
- metallic bonding: metals (~ 20 N/m)
- van der Waals interaction: polymers (~ 0.5 N/m)

Materials are made of many atoms, governed by thermodynamics

- materials choose structures, phase variables (such as density) that **minimize free energy**: $A = U - TS$
- A : Helmholtz free energy
- U : internal energy (bonding)
- T : (absolute) temperature
- S : entropy (disorder: $k_B \log \Omega$)

Thermodynamic "equation of state"



$$P(V) = \left. \frac{\partial A}{\partial V} \right|_V$$

Universality of linear elastic response

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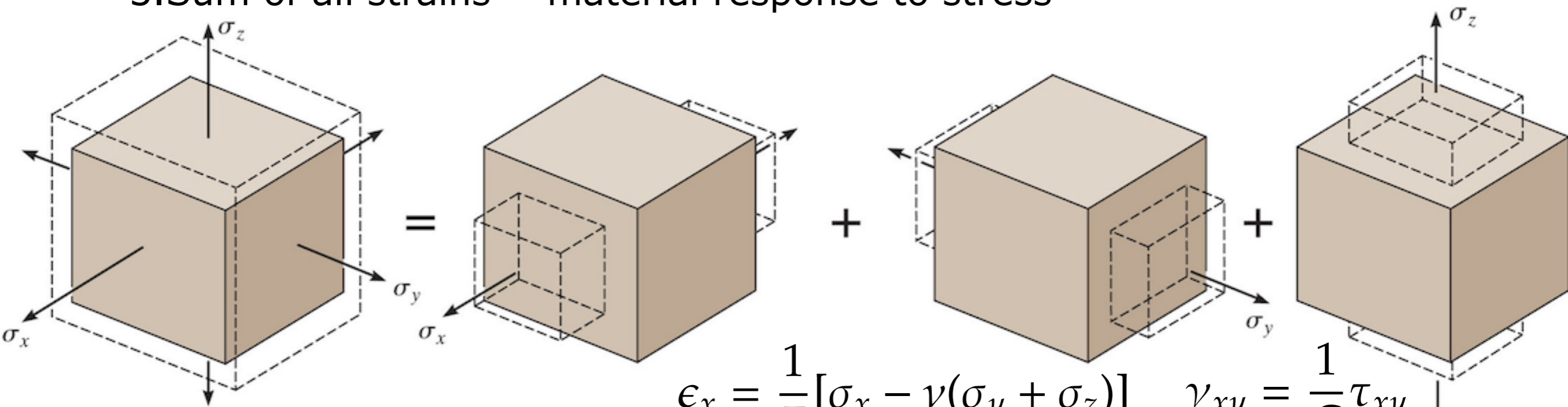
$$\begin{aligned} P(V) &= \left. \frac{\partial A}{\partial V} \right|_V \\ P(\delta V + V_0) &= \left. \frac{\partial A}{\partial V} \right|_{V_0} + \delta V \left. \frac{\partial^2 A}{\partial V^2} \right|_{V_0} + \frac{1}{2} \delta V^2 \left. \frac{\partial^3 A}{\partial V^3} \right|_{V_0} + \dots \\ &= 0 + \frac{\delta V}{V_0} \left(V_0 \left. \frac{\partial^2 A}{\partial V^2} \right|_{V_0} \right) + \dots \\ &= \epsilon_V K \end{aligned}$$

Superposition principle

- For **small stresses** the strains are linearly related to stresses:

$$\epsilon_{\parallel} = \frac{1}{E}\sigma_{\parallel} \quad \epsilon_{\perp} = -\frac{\nu}{E}\sigma_{\parallel} \quad \gamma = \frac{1}{G}\tau$$

- We can generalize these results by considering **superposition**
 - Each stress component (σ_x σ_y σ_z T_{xy} T_{xz} T_{yz}) is considered individually
 - All of the strains from each stress component computed
 - Sum of all strains = material response to stress



$$\epsilon_x = \frac{1}{E}[\sigma_x - \nu(\sigma_y + \sigma_z)]$$

$$\gamma_{xy} = \frac{1}{G}\tau_{xy}$$

$$\epsilon_y = \frac{1}{E}[\sigma_y - \nu(\sigma_x + \sigma_z)]$$

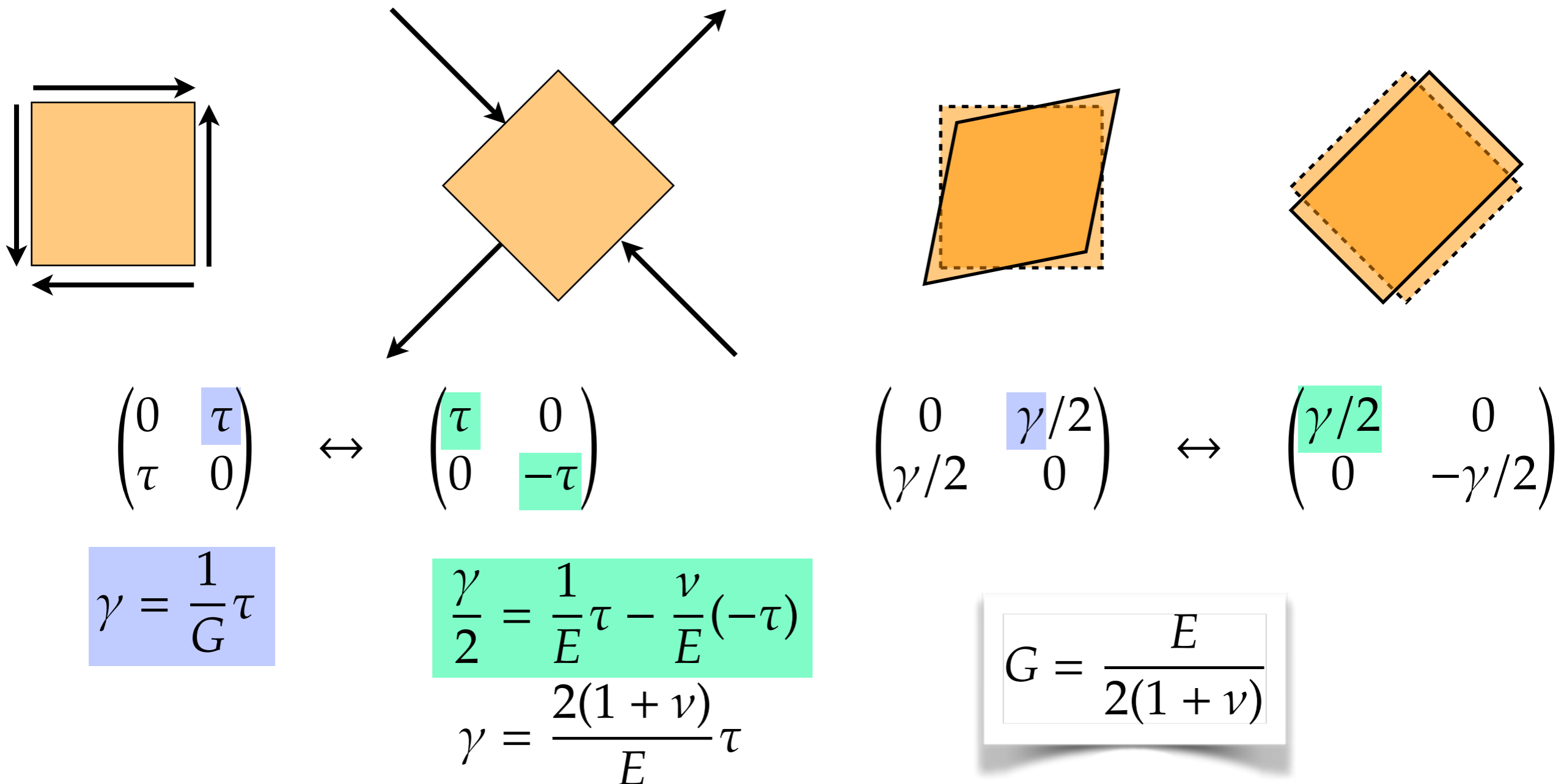
$$\gamma_{xz} = \frac{1}{G}\tau_{xz}$$

$$\epsilon_z = \frac{1}{E}[\sigma_z - \nu(\sigma_x + \sigma_y)]$$

$$\gamma_{yz} = \frac{1}{G}\tau_{yz}$$

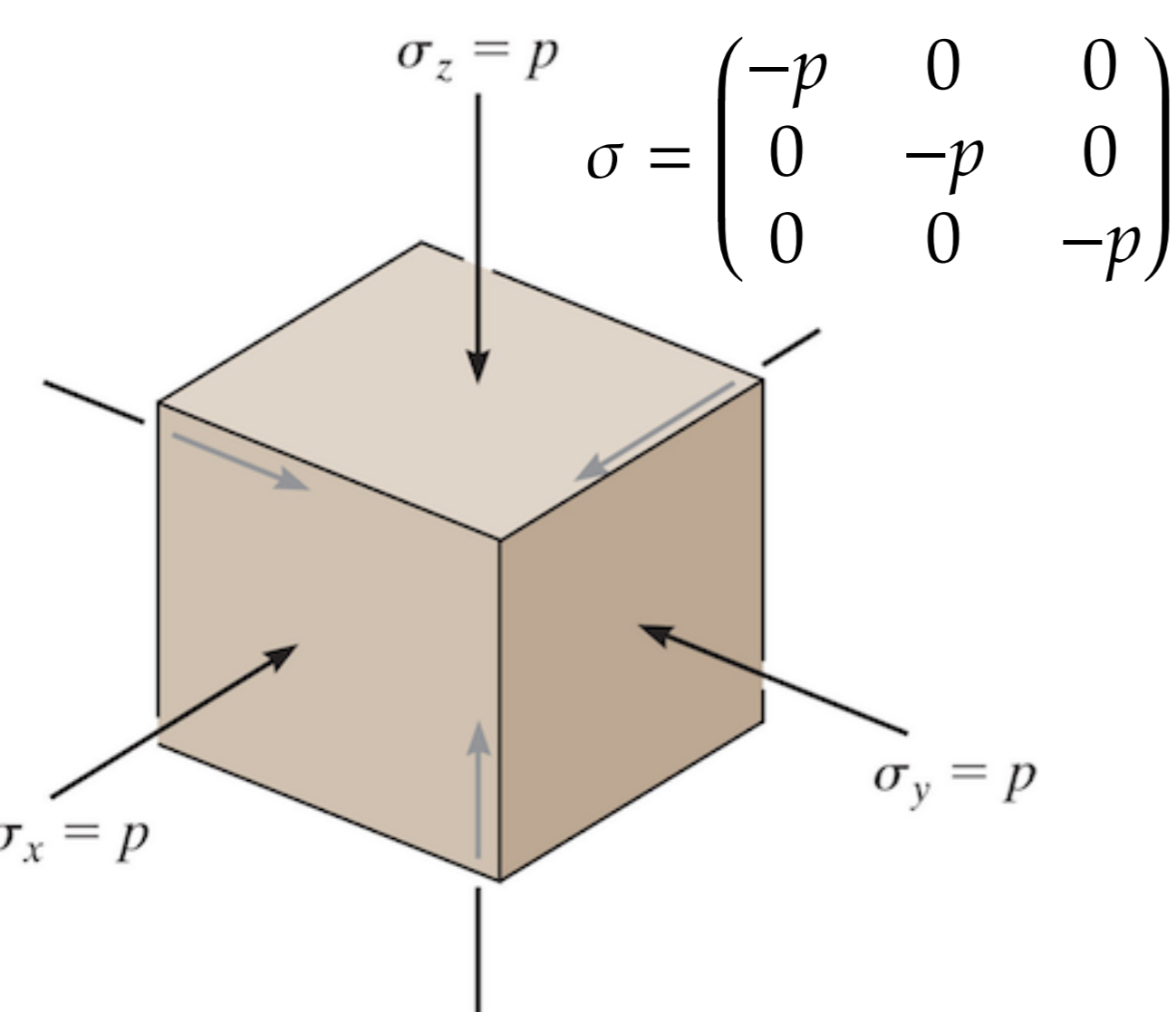
Material property relationships

- The superposition principle can relate our elastic moduli:
 - E : Young's modulus (normal strain from uniaxial stress)
 - ν : Poisson's ratio (perpendicular normal strain from uniaxial stress)
 - G : shear modulus (shear strain from shear stress)
 - K : bulk modulus (volume change from hydrostatic pressure)



Material property relationships

- The superposition principle can relate our elastic moduli:
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 - G : shear modulus (shear strain from shear stress)
 - K : bulk modulus (volume change from hydrostatic pressure)



$$\sigma = \begin{pmatrix} -p & 0 & 0 \\ 0 & -p & 0 \\ 0 & 0 & -p \end{pmatrix} \quad \epsilon = \begin{pmatrix} -\frac{p}{E} + 2\nu\frac{p}{E} & 0 & 0 \\ 0 & -\frac{p}{E} + 2\nu\frac{p}{E} & 0 \\ 0 & 0 & -\frac{p}{E} + 2\nu\frac{p}{E} \end{pmatrix}$$

$$\begin{aligned} \Delta V &= V(1 + \epsilon_x)(1 + \epsilon_y)(1 + \epsilon_z) - V \\ &= V(1 + (\epsilon_x + \epsilon_y + \epsilon_z) + \dots) - V \end{aligned}$$

$$\frac{\Delta V}{V} \approx \epsilon_x + \epsilon_y + \epsilon_z$$

$$= -\frac{3(1 - 2\nu)}{E}p = -\frac{p}{K}$$

$$K = \frac{E}{3(1 - 2\nu)}$$

Anisotropic linear elastic response

Isotropic stress/strain relations

$$\epsilon_x = \frac{1}{E} [\sigma_x - \nu(\sigma_y + \sigma_z)]$$

$$\epsilon_y = \frac{1}{E} [\sigma_y - \nu(\sigma_x + \sigma_z)]$$

$$\epsilon_z = \frac{1}{E} [\sigma_z - \nu(\sigma_x + \sigma_y)]$$

$$\sigma_x = \frac{E}{(1 + \nu)(1 - 2\nu)} [(1 - \nu)\epsilon_x + \nu(\epsilon_y + \epsilon_z)]$$

$$\sigma_y = \frac{E}{(1 + \nu)(1 - 2\nu)} [(1 - \nu)\epsilon_y + \nu(\epsilon_x + \epsilon_z)]$$

$$\sigma_z = \frac{E}{(1 + \nu)(1 - 2\nu)} [(1 - \nu)\epsilon_z + \nu(\epsilon_x + \epsilon_y)]$$

$$\gamma_{xy} = \frac{1}{G} \tau_{xy}$$

$$\gamma_{xz} = \frac{1}{G} \tau_{xz}$$

$$\gamma_{yz} = \frac{1}{G} \tau_{yz}$$

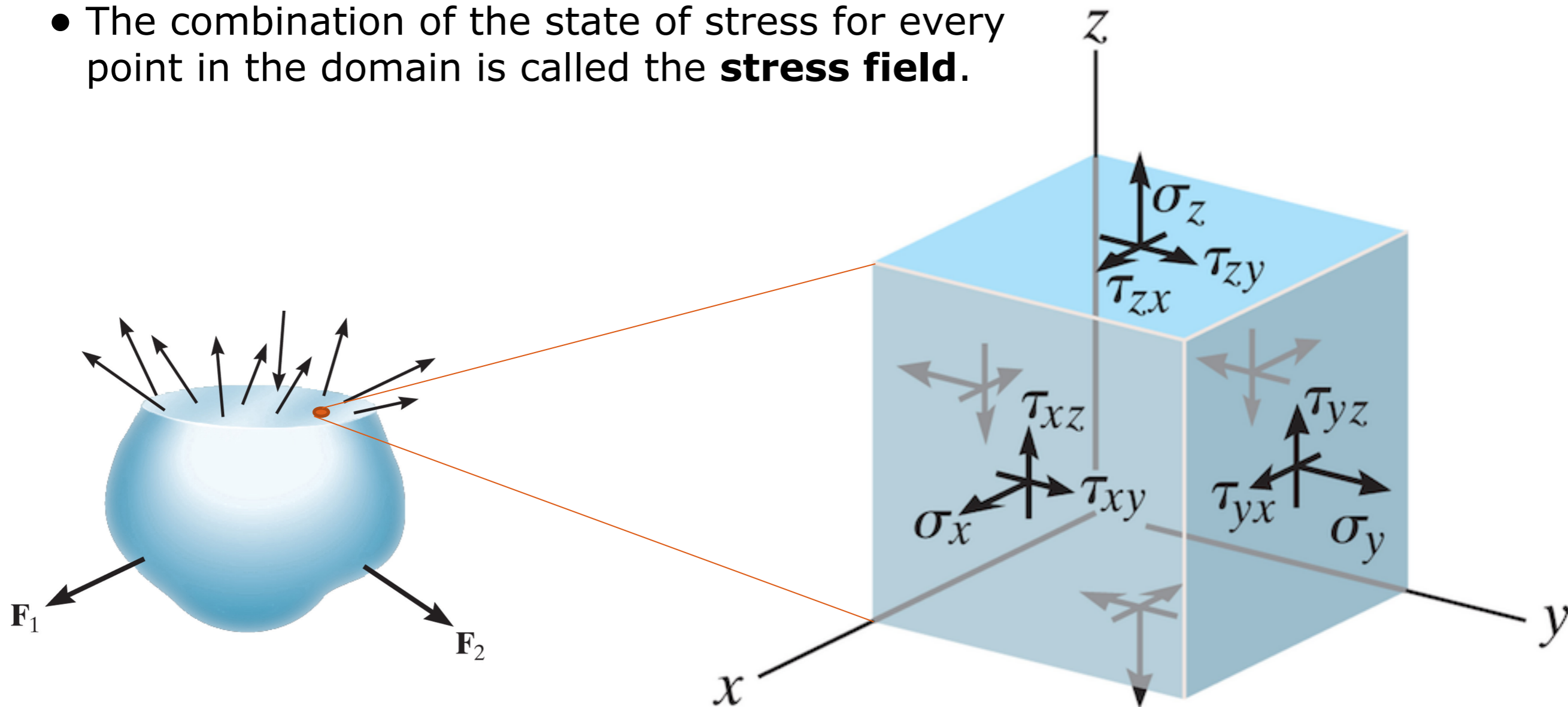
$$\tau_{xy} = G\gamma_{xy}$$

$$\tau_{xz} = G\gamma_{xz}$$

$$\tau_{yz} = G\gamma_{yz}$$

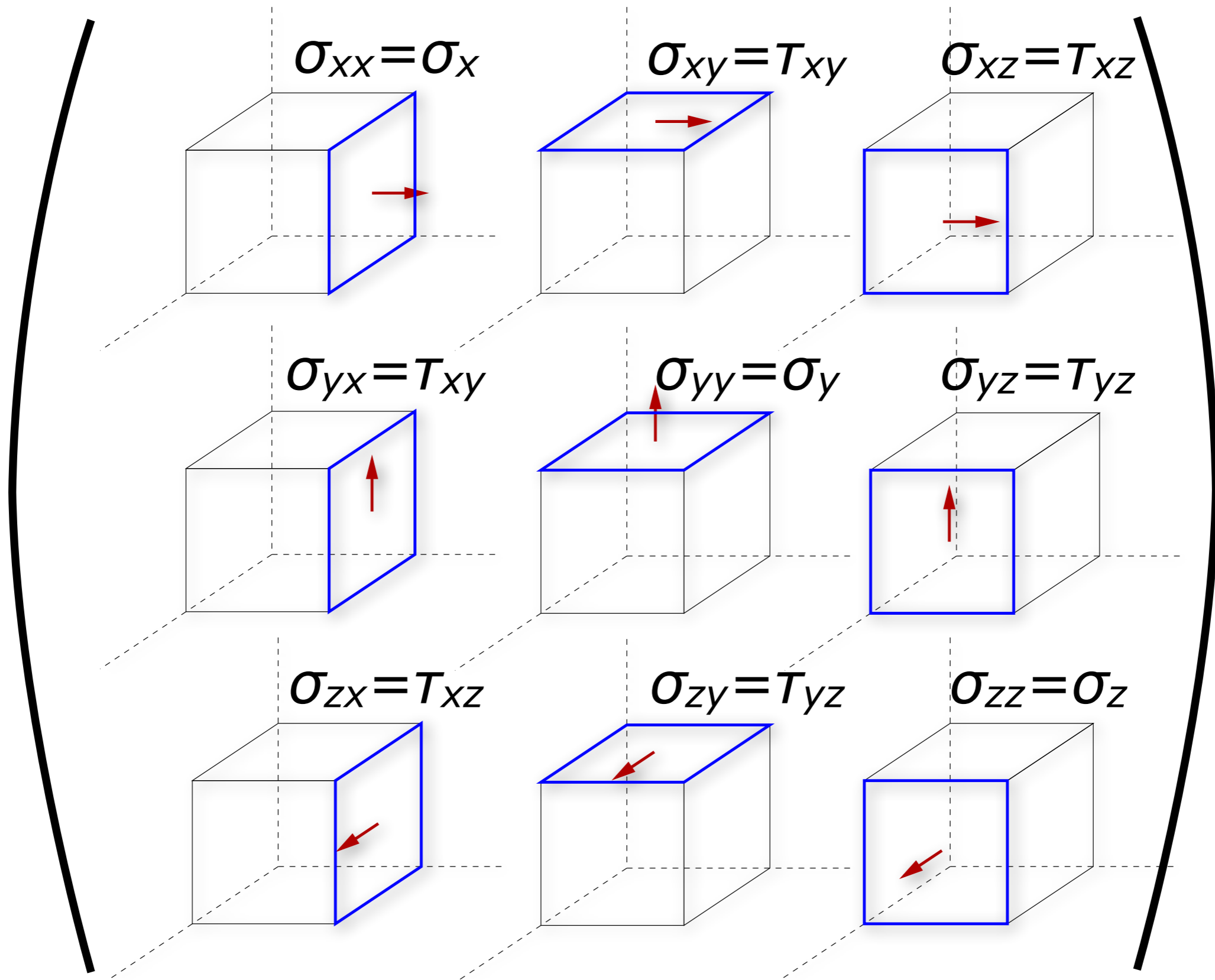
Is there a way to extend this to anisotropic response?

- Each point in a body has *normal* and *shear* stress components.
- We can section a *cubic volume* of material that represents the **state of stress** acting around the chosen point.
- As the cube is at equilibrium the **total forces and moments** are zero:
 - Infinitesimal cube = equal and opposite forces on opposite sides of cube
- Note also: the values of all the components depend on how the cube is oriented in the material (we'll talk later about relating those values)
- The combination of the state of stress for every point in the domain is called the **stress field**.



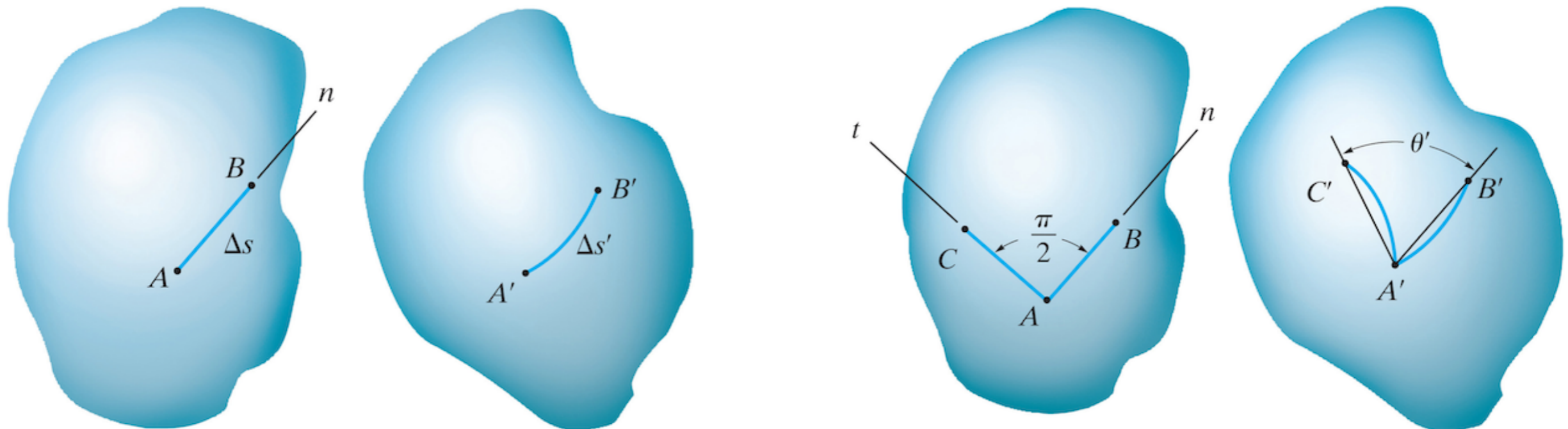
Graphical stress tensor components

- Stress \times area = force $F_i = \sum_{j=xyz} \sigma_{ij} A_j$



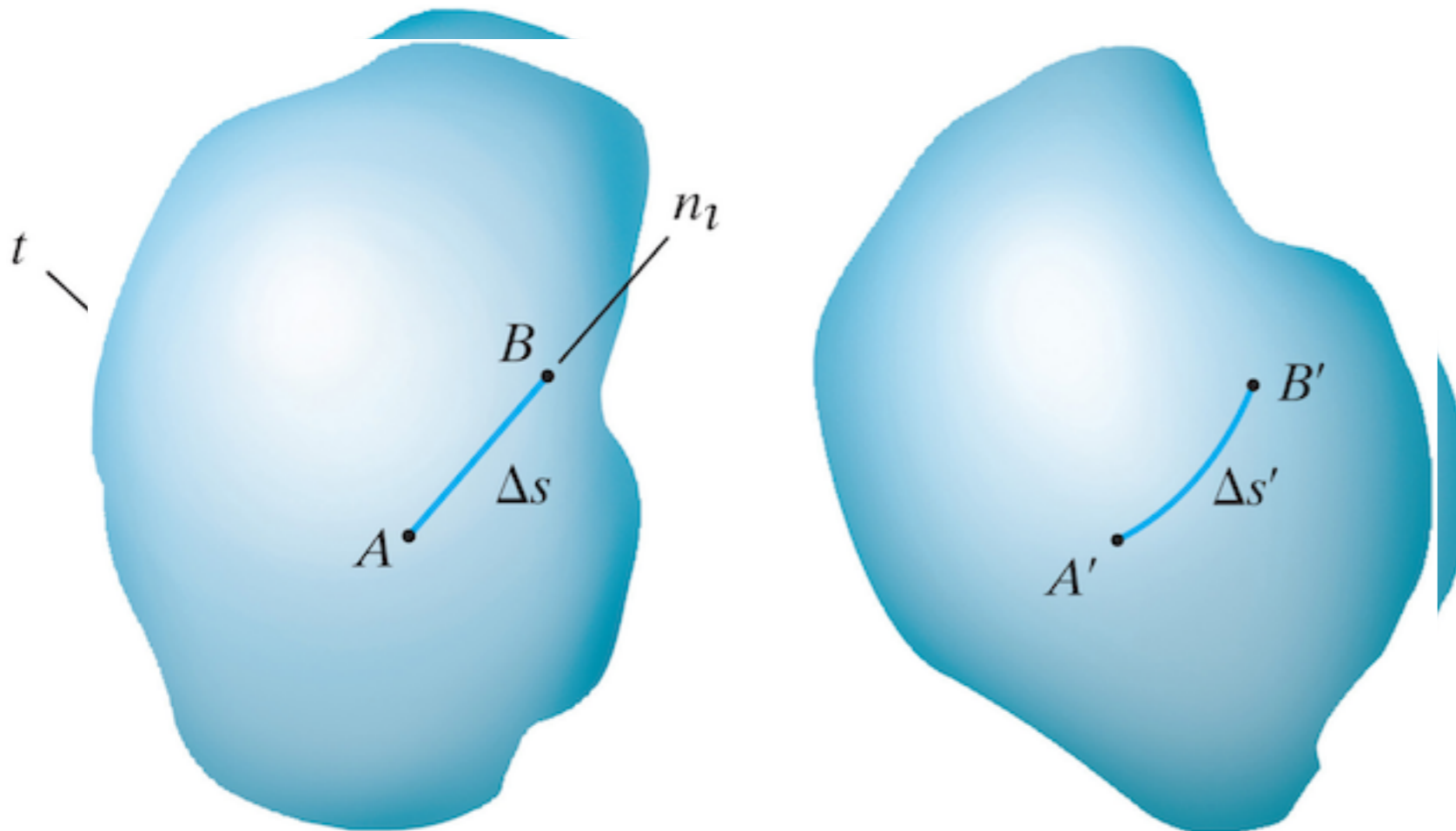
Defining strain

- We want to describe the *dimension and shape change* in a *continuous cohesive* body
- In a sufficiently small element, deformations of the element are all proportional to the size of the element
- length / length = unitless, % (10^{-2}), mm/mm, $\mu\text{m}/\text{m}$ (10^{-6}), or in/in
- Requires that we capture **both** the orientation of *original vector* and *change in that vector*
 - Original relative position is a *vector*: *one index* = 3 numbers to describe
 - New relative position is a *vector*: *one index* = 3 numbers to describe
 - Strain is a *tensor*: *two indices* (coordinates) = 3×3 numbers to describe



Normal strain and shear strain

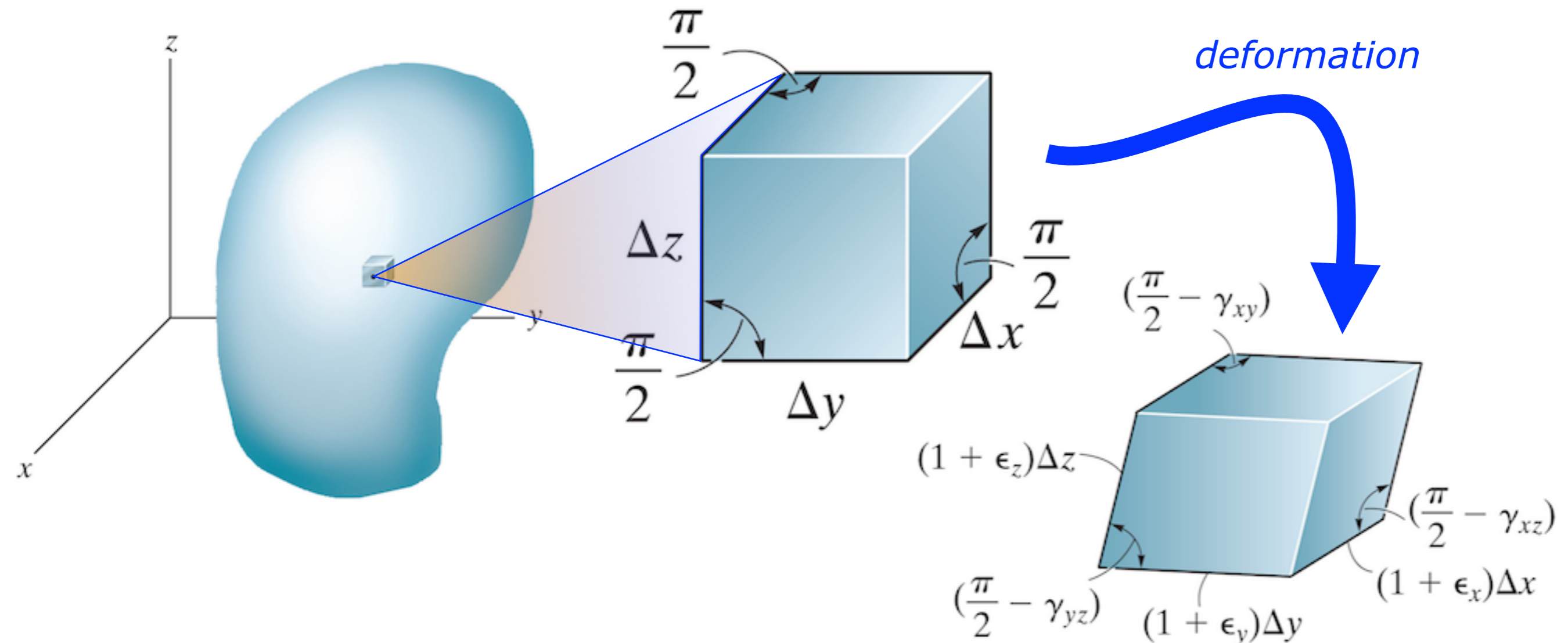
- Normal strain describes a length change in a vector
- Shear strain describes an orientation change in a vector
- **Be aware:** whether deformation *changes length* or *changes orientation* also depends on the *original orientation*



$$\epsilon \equiv \lim_{\Delta s \rightarrow 0} \frac{\Delta s' - \Delta s}{\Delta s}$$

Normal strain and shear strain

- We can describe all the strains on an *element* in a body:

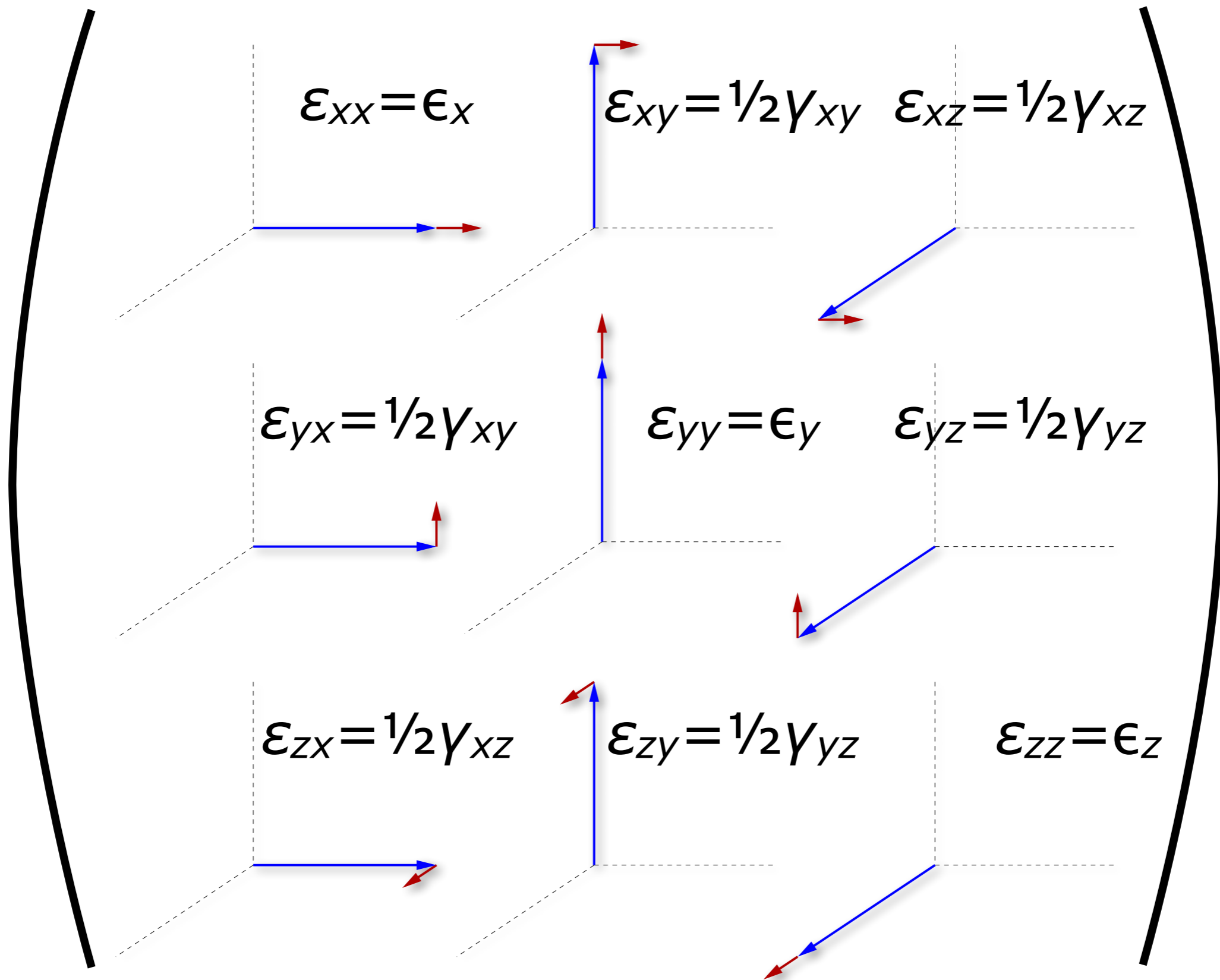


$$\epsilon_{ij} = \begin{pmatrix} \epsilon_x & \frac{1}{2}\gamma_{xy} & \frac{1}{2}\gamma_{xz} \\ \frac{1}{2}\gamma_{xy} & \epsilon_y & \frac{1}{2}\gamma_{yz} \\ \frac{1}{2}\gamma_{xz} & \frac{1}{2}\gamma_{yz} & \epsilon_z \end{pmatrix}$$

shear strains $\gamma_{xy} = (\text{angle change in } xy \text{ plane})$
 $= \epsilon_{xy} + \epsilon_{yx}$
 $\theta_{xy} = (\text{rotation in } xy \text{ plane})$
 $= \epsilon_{yx} - \epsilon_{xy}$
normal strains

Graphical strain tensor components

- Strain \times length = length change $\delta l_i = \sum_{j=x,y,z} \epsilon_{ij} l_j$



Elastic constants: stiffnesses and compliances 17

- Just as stress relates a vector (area) to another vector (force), and strain relates a vector (position) to another vector (change in position), our elastic constants relate **stresses** to **strains**: 4th rank tensors

$$\epsilon_{ij} = \sum_{kl} S_{ijkl} \sigma_{kl}$$

compliance
[GPa⁻¹]

$$\sigma_{ij} = \sum_{kl} C_{ijkl} \epsilon_{kl}$$

stiffness
[GPa]

- 3×3×3×3 = 81 components!
- But first two and last two are **symmetric**: $xyzz = yxzz$ and $zzxy = zzyx$
- And first pair and last pair can be swapped: $xyzz = zzyx$
 - Stiffness is a second derivative of energy: $C_{ijkl} = d^2U/d\epsilon_{ij} d\epsilon_{kl}$
- Results in 21 unique elastic constants. Better written with Voigt notation:

$$\begin{pmatrix} \sigma_1 & \sigma_6 & \sigma_5 \\ \sigma_6 & \sigma_2 & \sigma_4 \\ \sigma_5 & \sigma_4 & \sigma_3 \end{pmatrix}$$

$$\begin{pmatrix} e_1 & \frac{1}{2}e_6 & \frac{1}{2}e_5 \\ \frac{1}{2}e_6 & e_2 & \frac{1}{2}e_4 \\ \frac{1}{2}e_5 & \frac{1}{2}e_4 & e_3 \end{pmatrix}$$

$$e_i = \sum_{j=1}^6 S_{ij} \sigma_j$$

$$\sigma_i = \sum_{j=1}^6 C_{ij} e_j$$

1	2	3
xx	yy	zz
4	5	6
yz	xz	xz

Elastic constants: stiffnesses and compliances 18

- The S and C matrices are **inverses** of each other
- The 21 stiffness and compliance matrix entries have factors of 2 and 4 to convert to tensor components:
 - $C_{ab} = C_{ijkl}$ for $a=1..6, b=1..6$
 - $S_{ab} = S_{ijkl}$ for $a=1..3$ and $b=1..3$
 - $S_{ab} = 2S_{ijkl}$ for $a=1..3$ and $b=1..6$ or $a=4..6$ and $b=1..3$ or
 - $S_{ab} = 4S_{ijkl}$ for $a=4..6$ and $b=4..6$
- Crystalline symmetry reduces the number of unique and nonzero entries

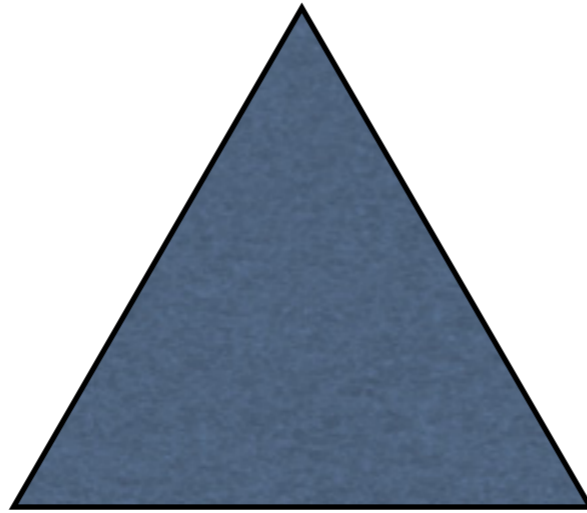
Stiffness / Compliance symmetry

$$\left(\begin{array}{cccccc}
 xx|xx & \begin{pmatrix} xx|yy \\ yy|xx \end{pmatrix} & \begin{pmatrix} xx|zz \\ zz|xx \end{pmatrix} & \begin{pmatrix} xx|yz & xx|zy \\ yz|xx & zy|xx \end{pmatrix} & \begin{pmatrix} xx|zx & xx|xz \\ zx|xx & xz|xx \end{pmatrix} & \begin{pmatrix} xx|xy & xx|yx \\ xy|xx & yx|xx \end{pmatrix} \\
 \cdot & yy|yy & \begin{pmatrix} yy|zz \\ zz|yy \end{pmatrix} & \begin{pmatrix} yy|yz & yy|zy \\ yz|yy & zy|yy \end{pmatrix} & \begin{pmatrix} yy|zx & yy|xz \\ zx|yy & xz|yy \end{pmatrix} & \begin{pmatrix} yy|xy & yy|yx \\ xy|yy & yx|yy \end{pmatrix} \\
 \cdot & \cdot & zz|zz & \begin{pmatrix} zz|yz & zz|zy \\ yz|zz & zy|zz \end{pmatrix} & \begin{pmatrix} zz|zx & zz|xz \\ zx|zz & xz|zz \end{pmatrix} & \begin{pmatrix} zz|xy & zz|yx \\ xy|zz & yx|zz \end{pmatrix} \\
 \cdot & \cdot & \cdot & \begin{pmatrix} yz|yz & yz|zy \\ zy|yz & zy|zy \end{pmatrix} & \begin{pmatrix} yz|zx & yz|xz & zx|yz & xz|yz \\ zy|zx & zy|xz & zx|zy & xz|zy \end{pmatrix} & \begin{pmatrix} yz|xy & yz|yx & xy|yz & yx|yz \\ zy|xy & zy|yx & xy|zy & yx|zy \end{pmatrix} \\
 \cdot & \cdot & \cdot & \cdot & \begin{pmatrix} zx|zx & zx|xz \\ xz|zx & xz|xz \end{pmatrix} & \begin{pmatrix} zx|xy & zx|yx & xy|zx & yx|zx \\ xz|xy & xz|yx & xy|xz & yx|xz \end{pmatrix} \\
 \cdot & \cdot & \cdot & \cdot & \cdot & \begin{pmatrix} xy|xy & xy|yx \\ yx|xy & yx|yx \end{pmatrix}
 \end{array} \right)$$

Symmetry operations



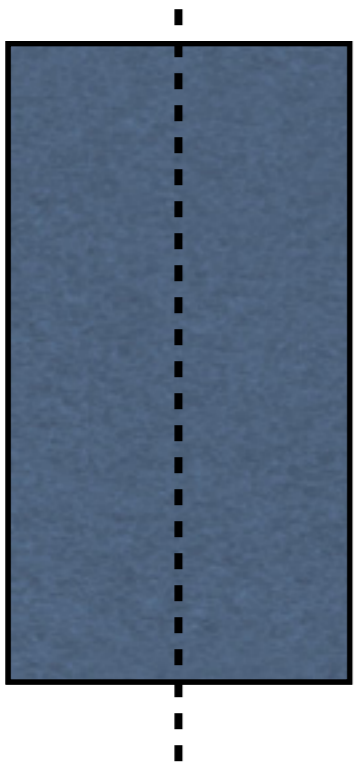
2-fold axis



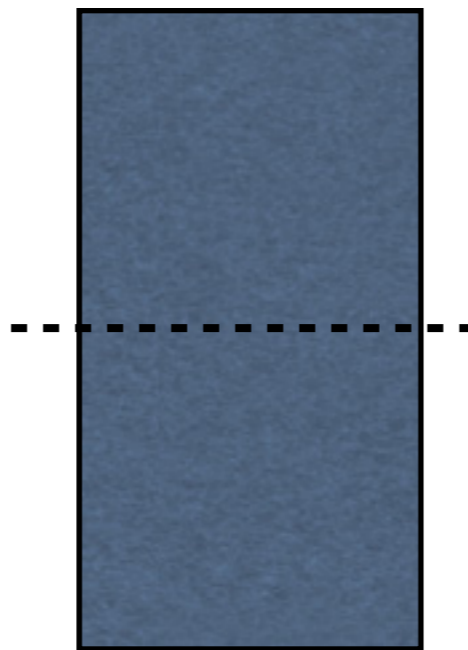
3-fold axis



4-fold axis

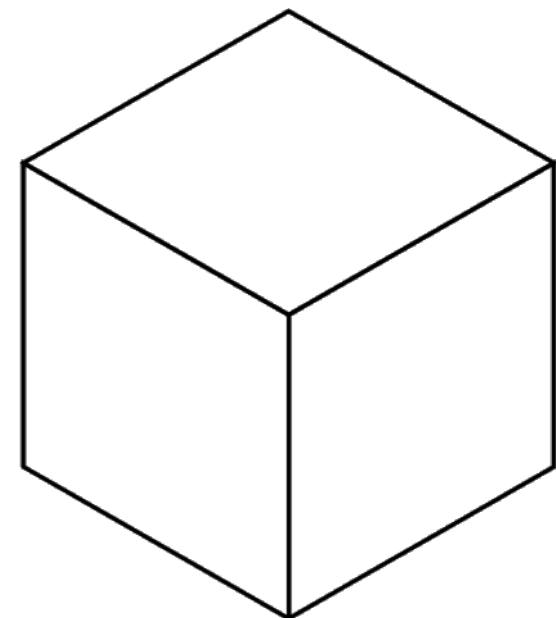


mirror plane



mirror plane

Rotating a cube around the
body diagonal $\langle 111 \rangle$?



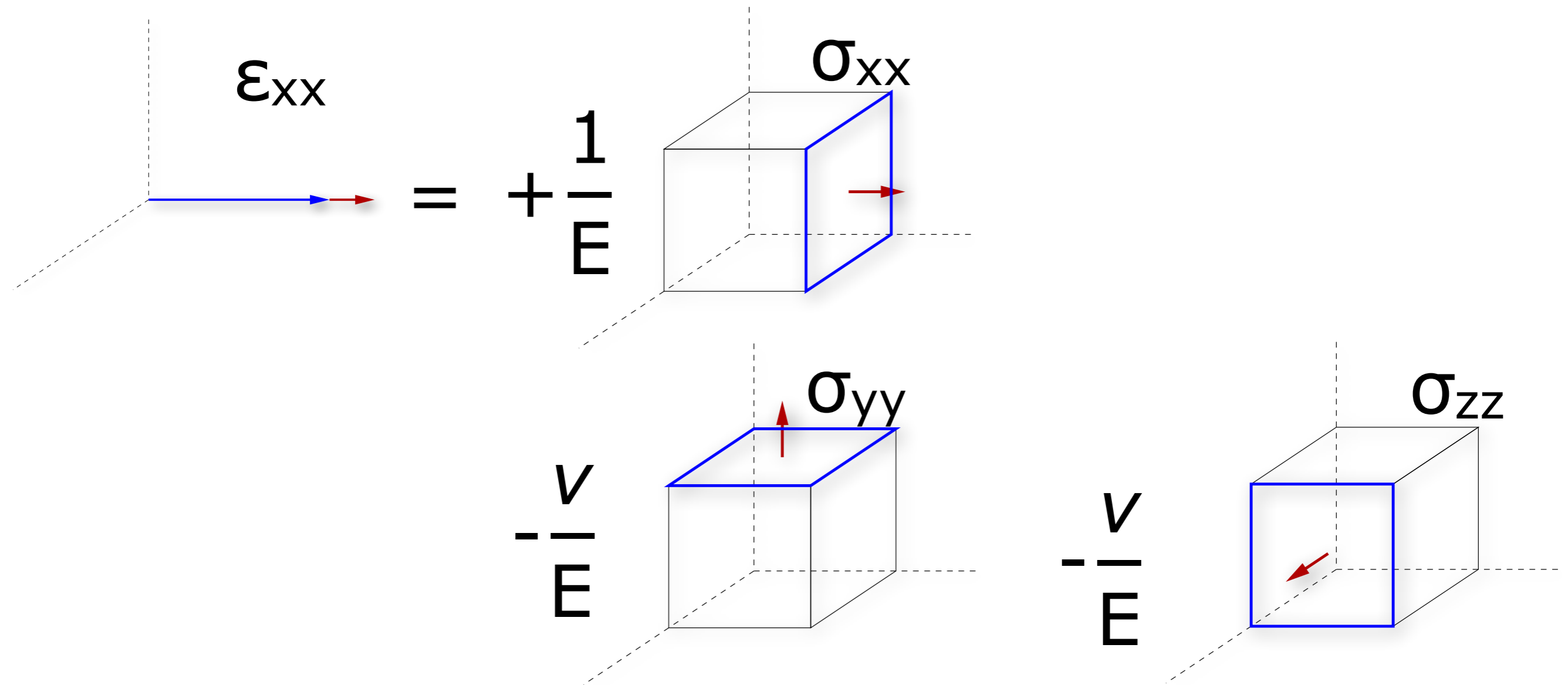
3-fold axis

Elastic constants: stiffnesses and compliances 21

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 - $S_{ab} = 4S_{ijkl}$ for $a=4..6$ and $b=4..6$
- Crystalline symmetry reduces the number of unique and nonzero entries
- **Cubic** symmetry is the most common for structural materials:
 - $C_{11} = C_{22} = C_{33}$
 - $C_{12} = C_{13} = C_{23}$
 - $C_{44} = C_{55} = C_{66}$
 - all others zero
- Isotropic materials are **cubic** and $C_{11} - C_{12} = 2C_{44}$ (or $S_{11} - S_{12} = S_{44}/2$)
- Hexagonal materials and aligned fiber composites have lower symmetry:
 - $C_{11} = C_{22} \neq C_{33}; C_{12} \neq C_{13} = C_{23}; C_{44} = C_{55} \neq C_{66}$
 - Isotropic in basal plane: $2C_{66} = C_{11} - C_{12}$
 - all others zero

Graphical compliance components

$$\epsilon_{ij} = \sum_{k=xyz} \sum_{l=xyz} S_{ijkl} \sigma_{kl}$$



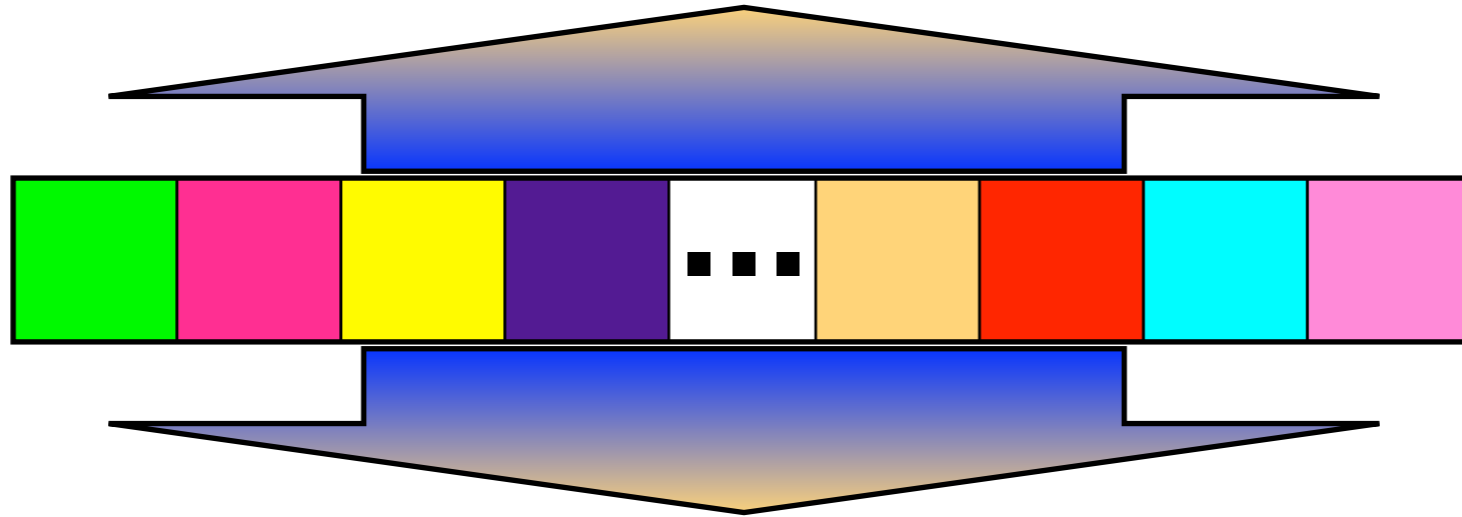
$$S_{xxxx} = 1/E \quad S_{xxyy} = -\nu/E \quad S_{xxzz} = -\nu/E$$

$$S_{xxkl} = 0 \text{ for all other } kl$$

Voigt and Reuss averages

Voigt average = **isostrain**

Randomly
oriented
grains



$$E_{\text{Voigt}} = \frac{(\bar{C}_{11} - \bar{C}_{12} + 3\bar{C}_{44})(\bar{C}_{11} + 2\bar{C}_{12})}{2\bar{C}_{11} + 3\bar{C}_{12} + \bar{C}_{44}}$$

$$\bar{C}_{11} = \frac{1}{3} (C_{11} + C_{22} + C_{33})$$

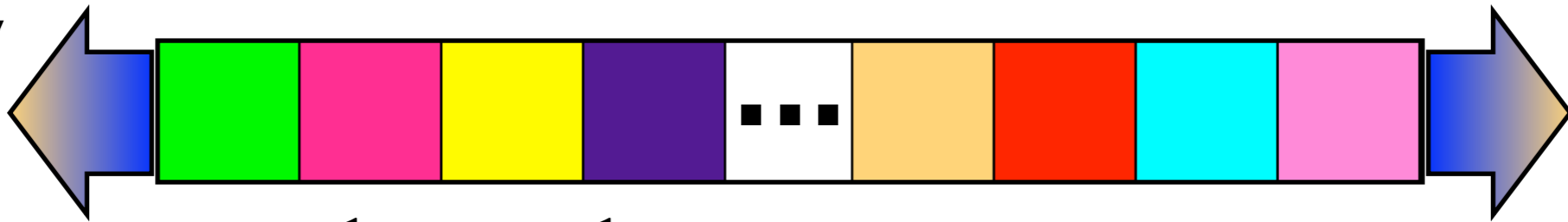
$$\bar{C}_{12} = \frac{1}{3} (C_{12} + C_{13} + C_{23})$$

$$\bar{C}_{44} = \frac{1}{3} (C_{44} + C_{55} + C_{66})$$

Voigt and Reuss averages

Reuss average = **isostress**

Randomly
oriented
grains



$$\frac{1}{E_{\text{Reuss}}} = \frac{1}{5} (3\bar{S}_{11} + 2\bar{S}_{12} + \bar{S}_{44})$$

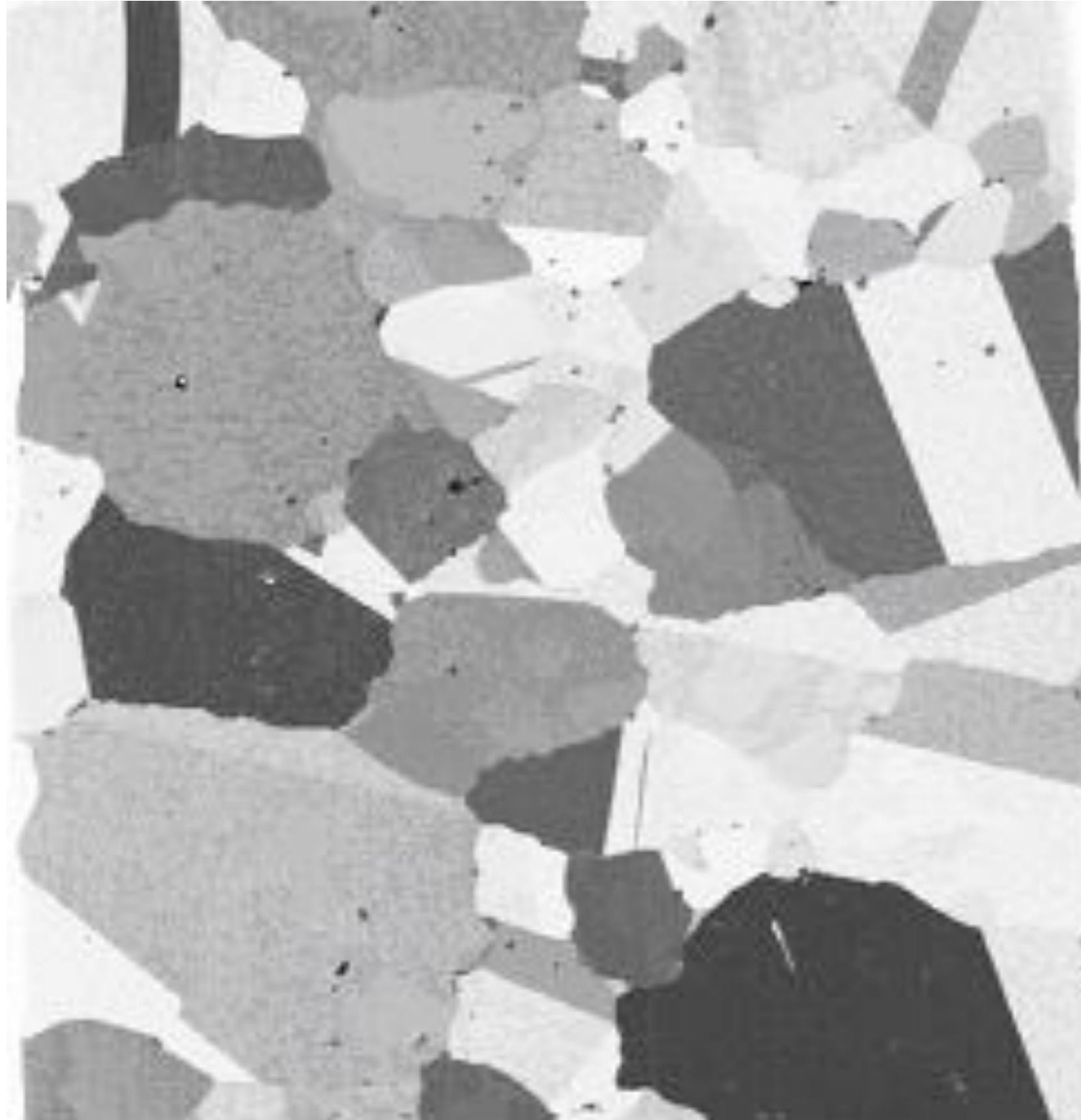
$$\bar{S}_{11} = \frac{1}{3} (S_{11} + S_{22} + S_{33})$$

$$\bar{S}_{12} = \frac{1}{3} (S_{12} + S_{13} + S_{23})$$

$$\bar{S}_{44} = \frac{1}{3} (S_{44} + S_{55} + S_{66})$$

$$E_{\text{Voigt}} > E_{\text{random polycrystal}} > E_{\text{Reuss}}$$

Grain structure and texture



Ni alloy grain structure

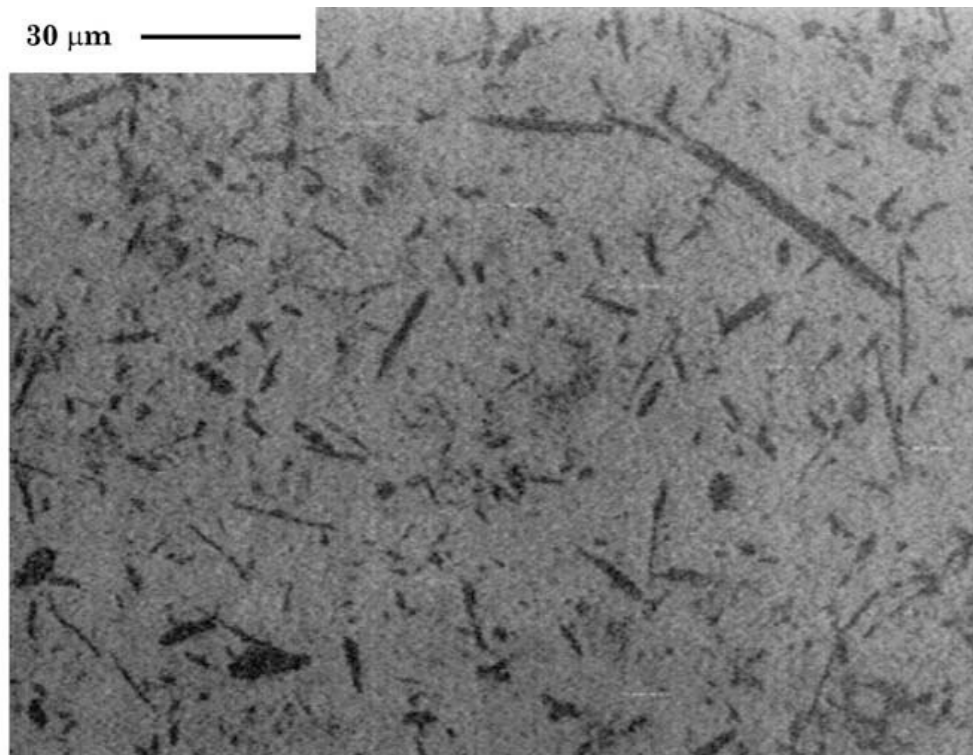
Each grain has a different orientation, and responds differently to applied stress

Polycrystalline response is an average of individual grain responses.

Texture is a preferential orientation of grains.

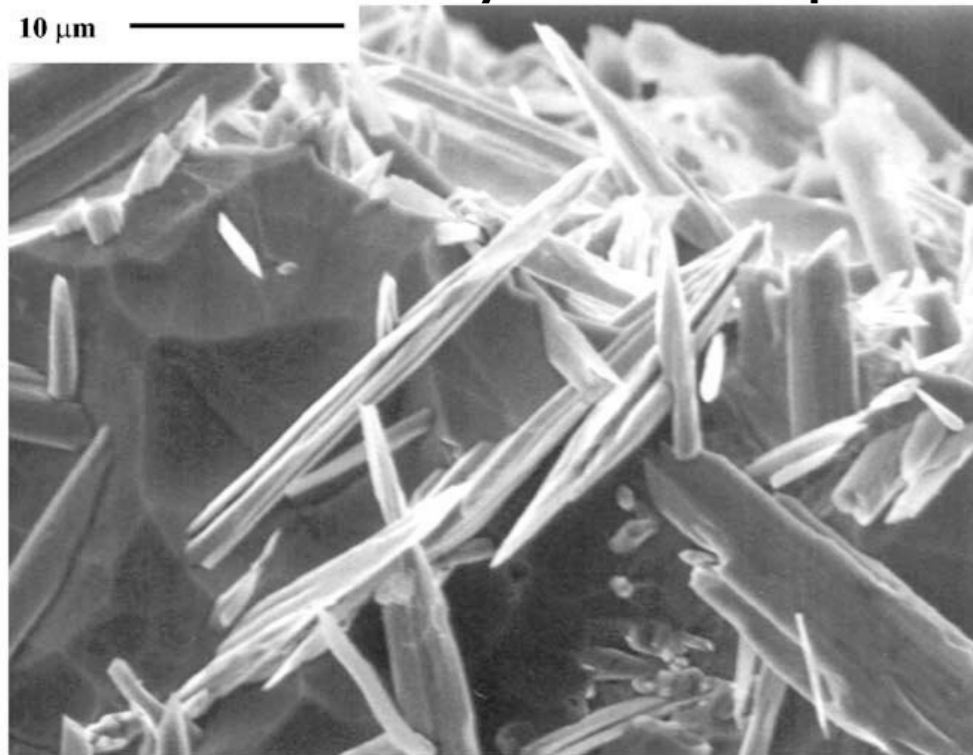
Ti / TiB metal-matrix composite

SEM backscatter: polish



(a)

SEM secondary e⁻: deep etch



(b)

TiB: orthorhombic crystal

$$C_{ij} = \begin{pmatrix} 419 & 92 & 113 & 0 & 0 & 0 \\ 92 & 523 & 63 & 0 & 0 & 0 \\ 113 & 63 & 418 & 0 & 0 & 0 \\ 0 & 0 & 0 & 196 & 0 & 0 \\ 0 & 0 & 0 & 0 & 179 & 0 \\ 0 & 0 & 0 & 0 & 0 & 220 \end{pmatrix} \text{ GPa}$$

$$E_{\text{Voigt}} = 442 \text{ GPa}$$

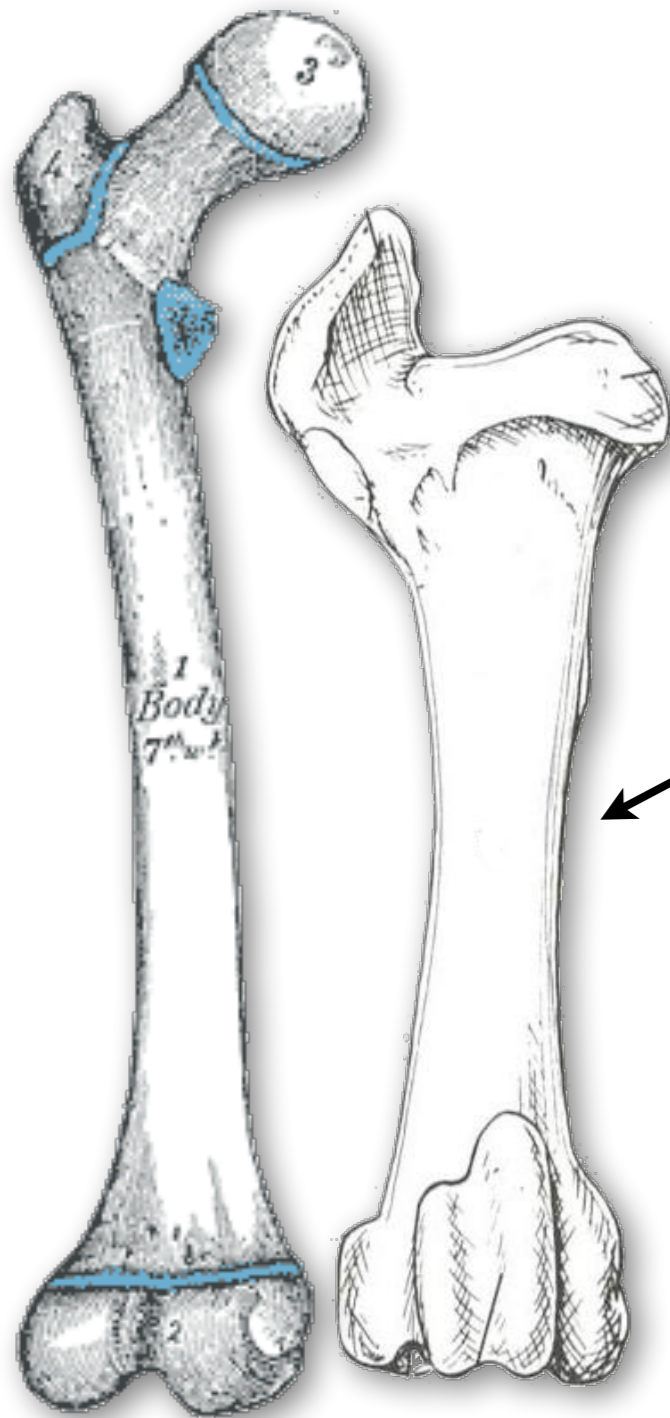
$$E_{\text{Reuss}} = 435 \text{ GPa}$$

$$E_{\text{Ti}} = 110 \text{ GPa}$$

$$E_{\text{Ti}+20\% \text{vol TiB}} = 153 \text{ GPa}$$

S. Gorsse et al., Mat. Sci. Eng. A **340**, 80-87 (2003)
D. R. Trinkle, Scripta Mater. **56**, 273-276 (2007)

Bovine femoral bone: elastic constants



longitudinal (3)

transverse (1,2)

$C_{ij} =$

bone orthotropic stiffnesses:

$$C_{ij} = \begin{pmatrix} 14 & 6.3 & 4.8 & 0 & 0 & 0 \\ 6.3 & 18.4 & 7.0 & 0 & 0 & 0 \\ 4.8 & 7.0 & 25 & 0 & 0 & 0 \\ 0 & 0 & 0 & 7.0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 6.3 & 0 \\ 0 & 0 & 0 & 0 & 0 & 5.3 \end{pmatrix} \text{ GPa}$$

along length: $E_3 = 21.7\text{GPa}$

transverse: $E_1 = 11.6\text{GPa}$

Extracted from sound-speed measurements:

$$C_{11} = \rho(v_{11})^2 \quad C_{22} = \rho(v_{22})^2 \quad C_{44} = \rho(v_{23})^2$$

Composite behavior

Composite = matrix + reinforcement

Matrix: continuous phase

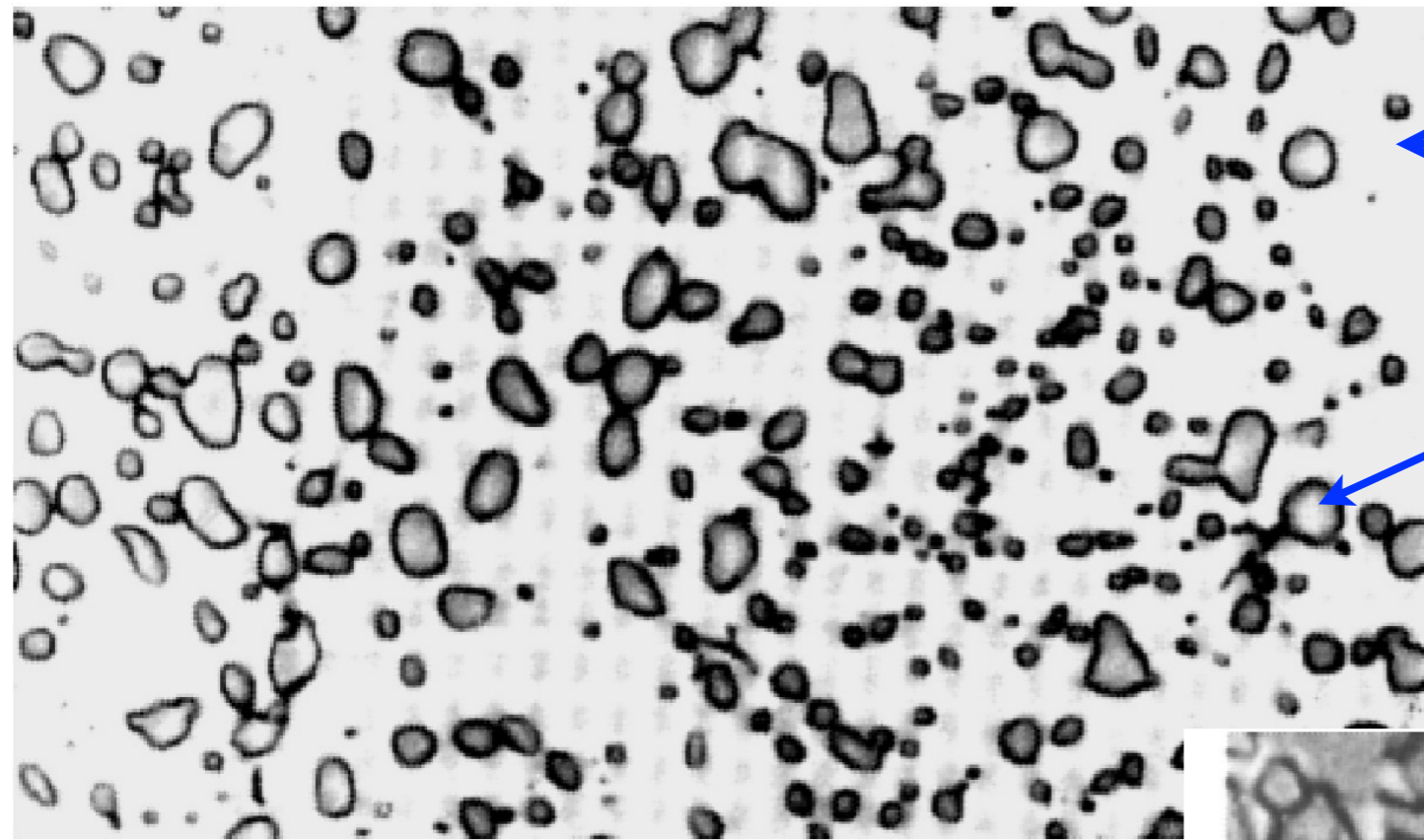
- transfers load to reinforcement
- protects reinforcement from environment

Types of matrix:

- **MMC** metal matrix composite: designed for **plastic strain**
 - better yield stress, tensile strength, creep resistance
- **CMC** ceramic matrix composite: designed for **fracture**
 - better toughness
- **PMC** polymer matrix composite: designed for **elastic and plastic strain**
 - better modulus, yield stress, tensile strength, creep
 - inexpensive, temperature range limited by polymer decomposition

Reinforcement: stronger, discontinuous phase

- carries significant portion of load
- classified by geometry

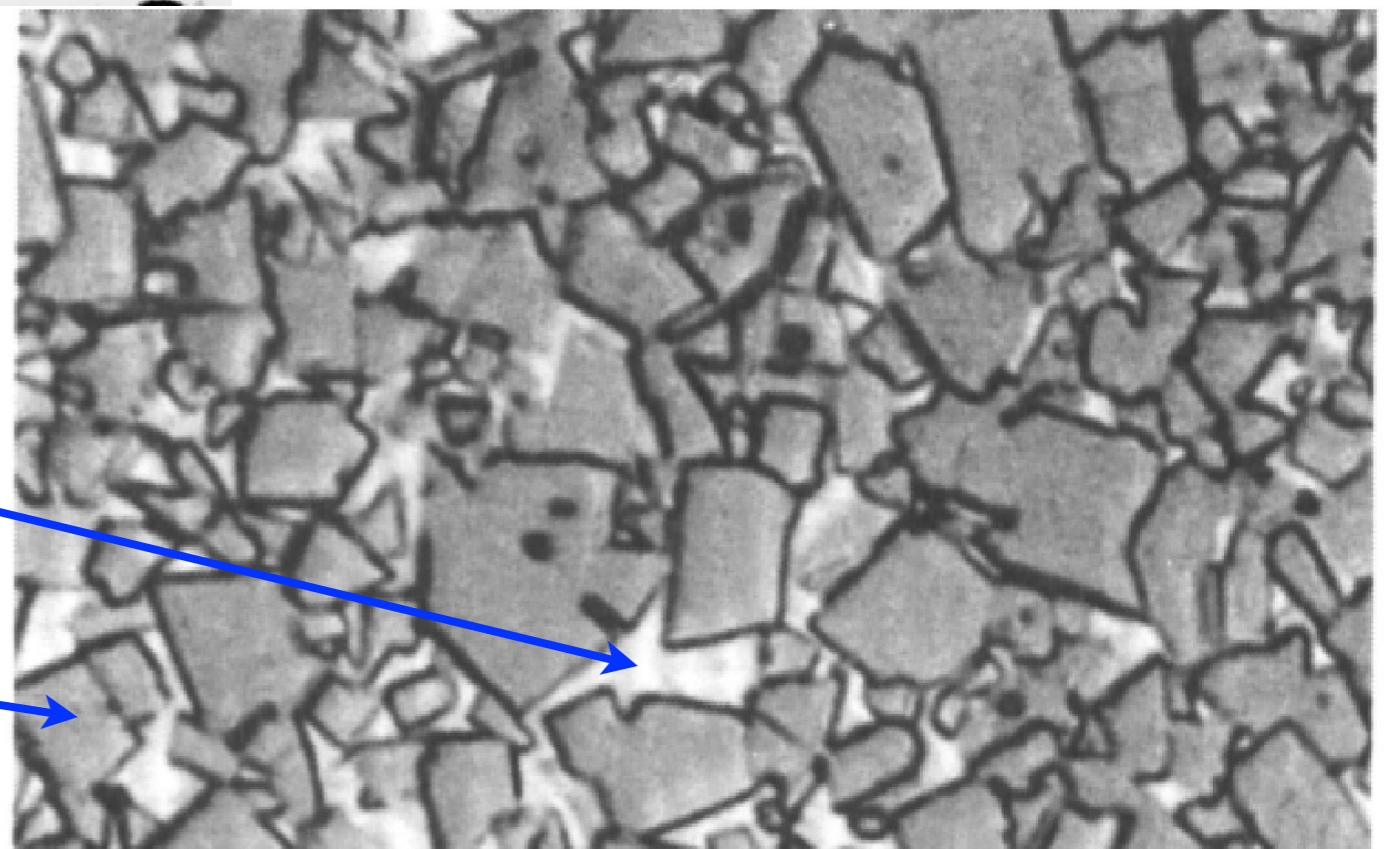


ferrite (bcc-Fe)

cementite (Fe_3C)

cemented carbide

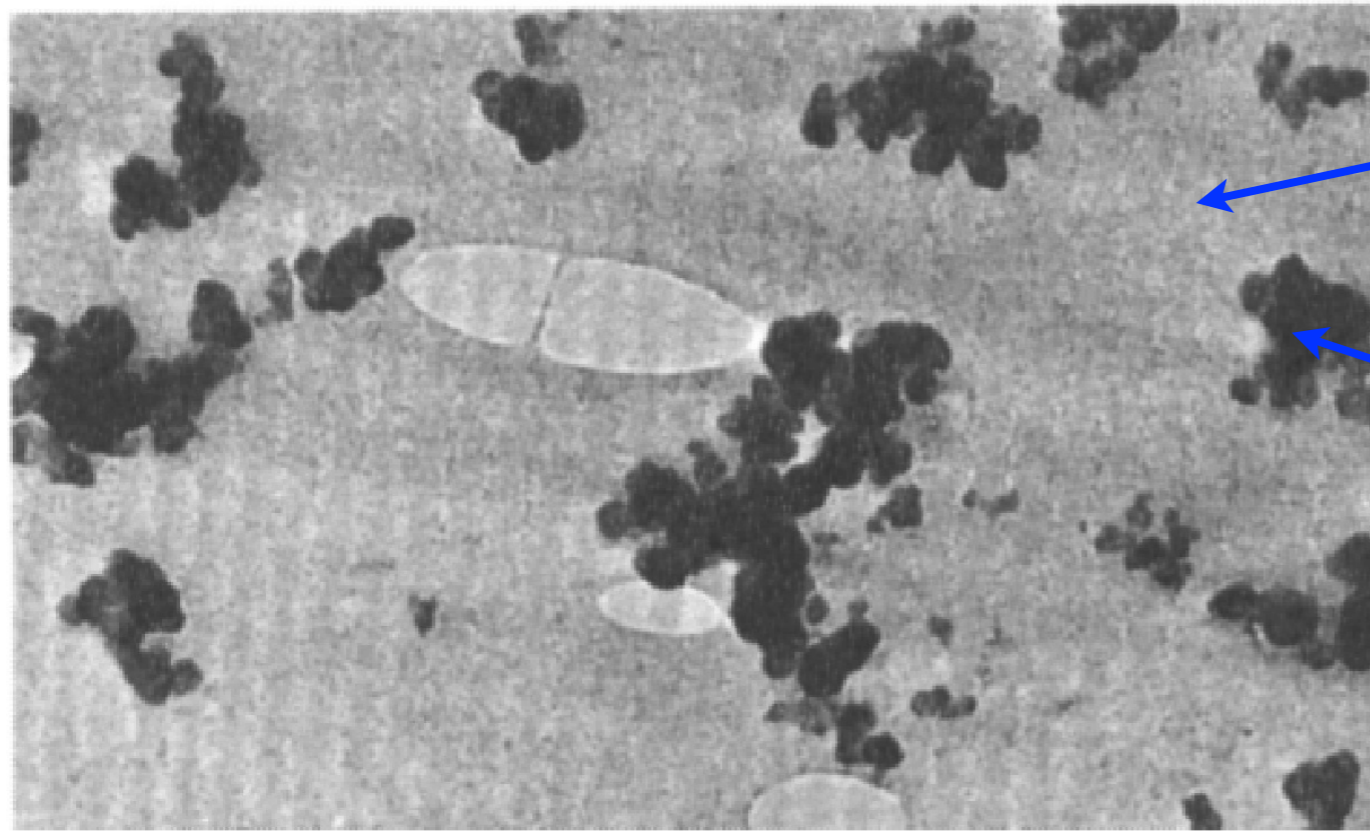
10 μm **Spheroidite steel**



Co matrix ($V_m=10-15\%$)

WC particles

100 μm

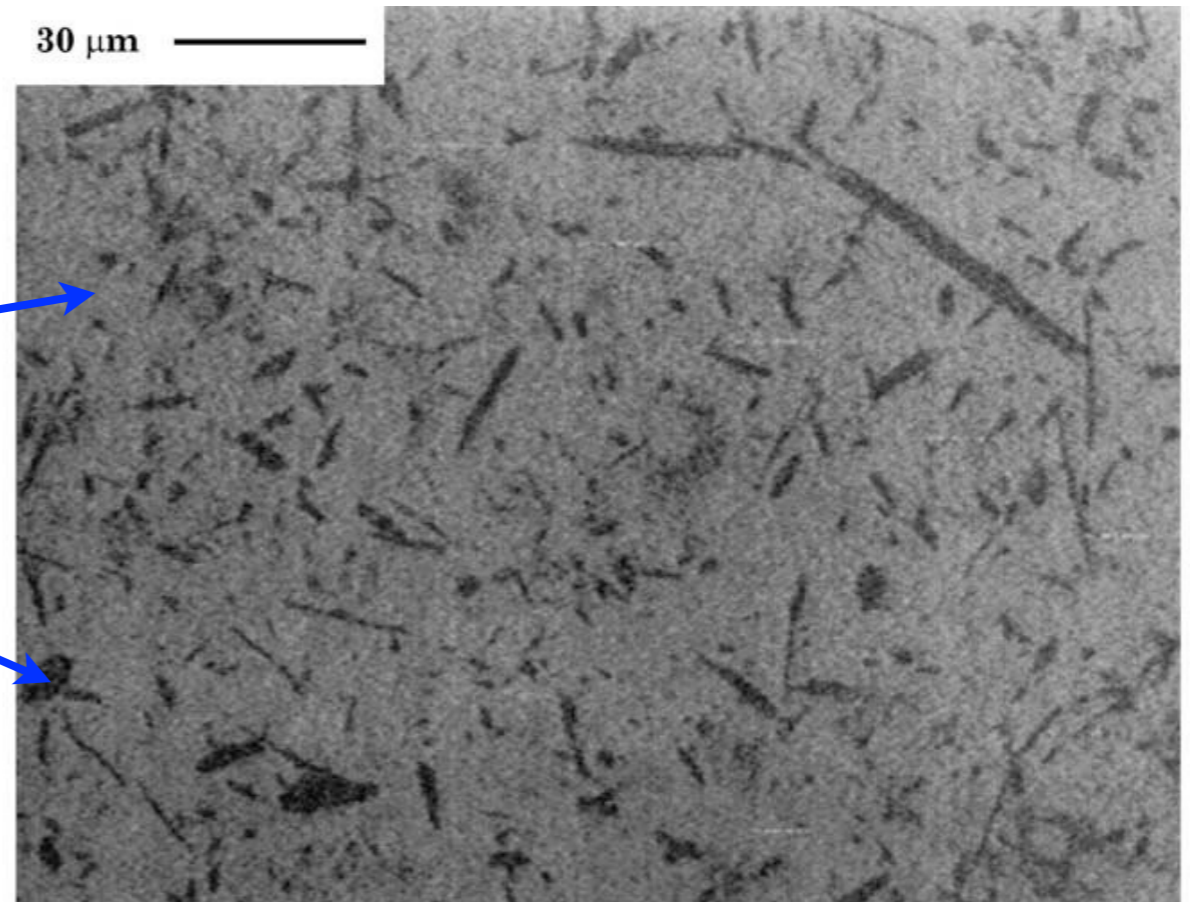


rubber

carbon particles

Ti/TiB MMC

30 μm



alpha-Ti (hcp)

TiB needles

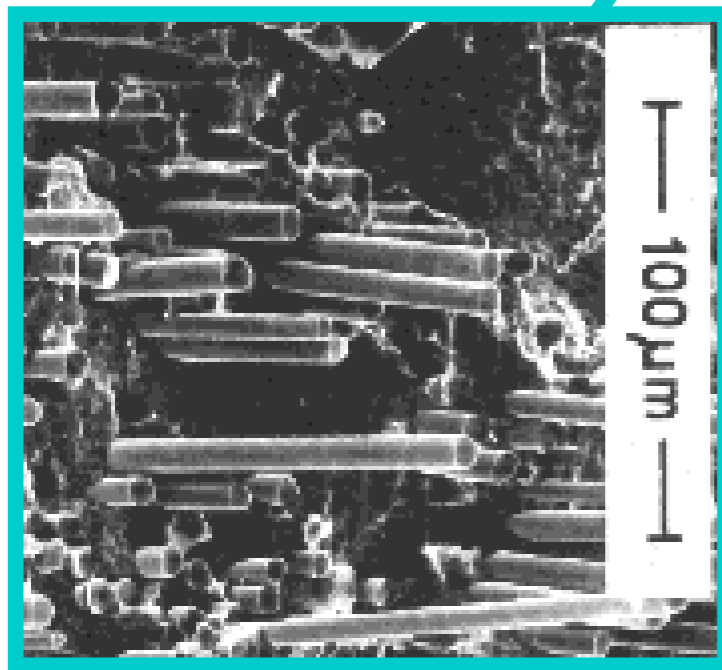
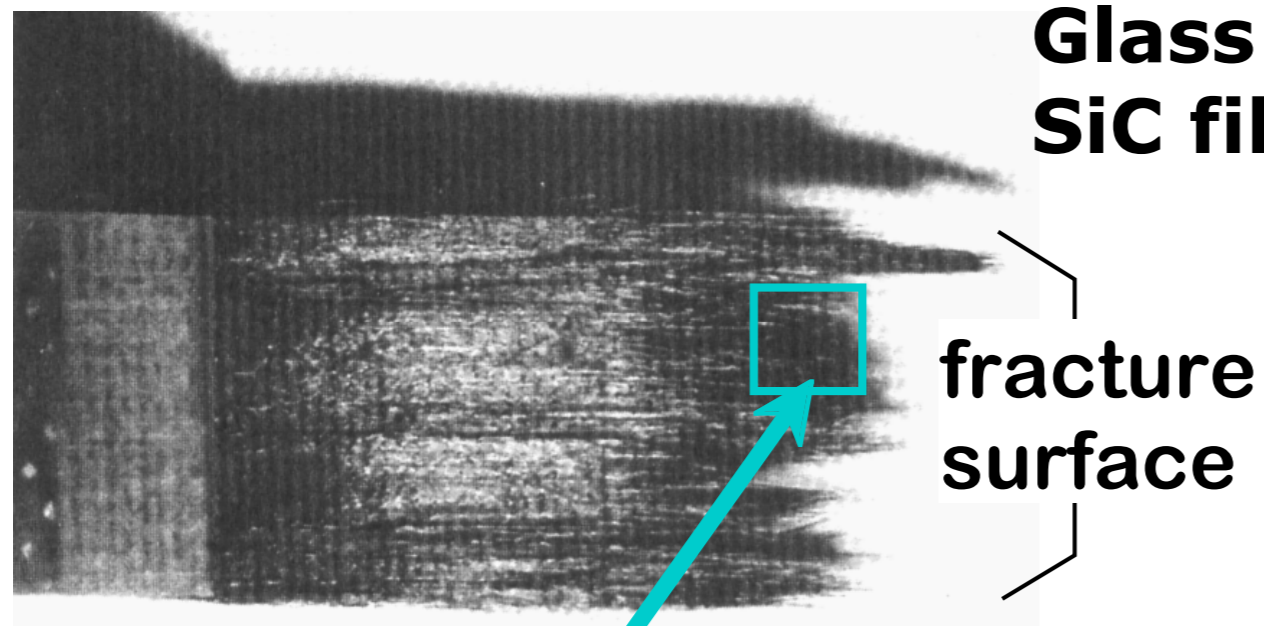
100nm

Tire rubber



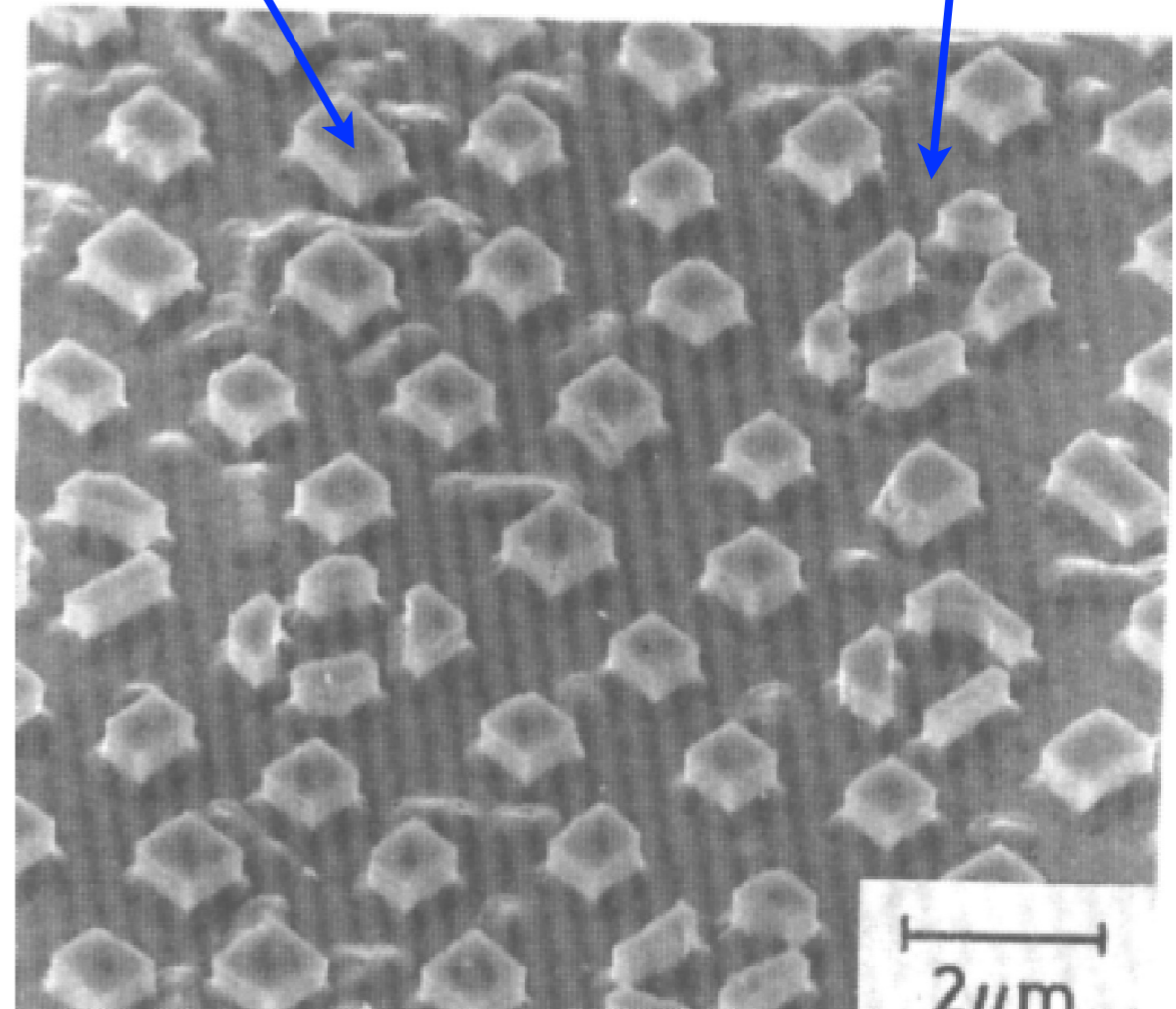
From Callister,
Intro to Eng. Matls., 6Ed

Fiber reinforcements: continuous, aligned



Ni_3Al (γ') $\alpha\text{-Mo}$ (bcc)

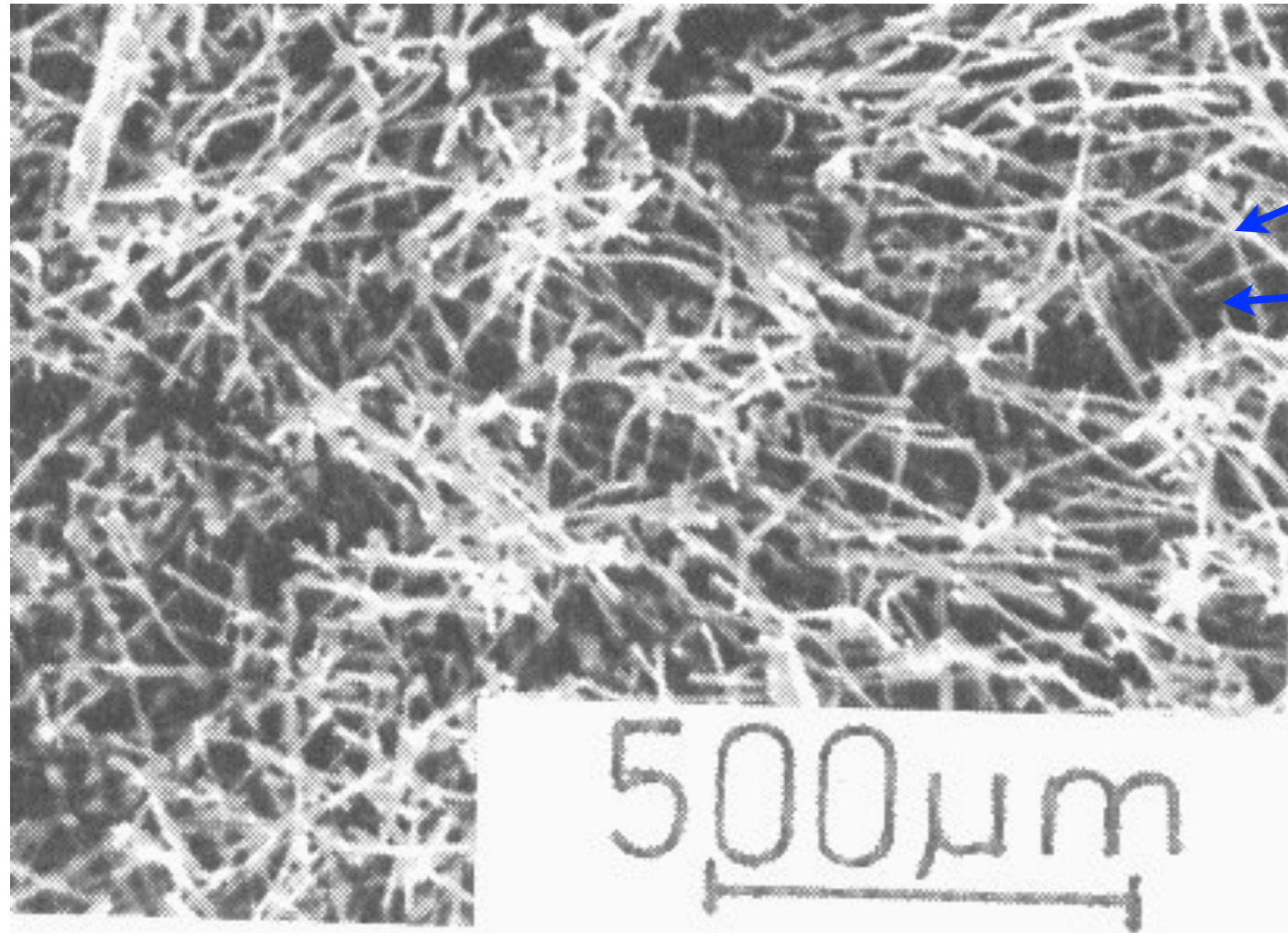
Mo + Ni_3Al



From F.L. Matthews and R.L. Rawlings, Composite Materials; Engineering and Science, Reprint ed., CRC Press, Boca Raton, FL, 2000. (a) Fig. 4.22, p. 145 (photo by J. Davies); (b) Fig. 11.20, p. 349 (micrograph by H.S. Kim, P.S. Rodgers, and R.D. Rawlings).

W. Funk et al., Met. Trans A19, 987-998 (1988).

Fiber reinforcements: discontinuous, random ³³



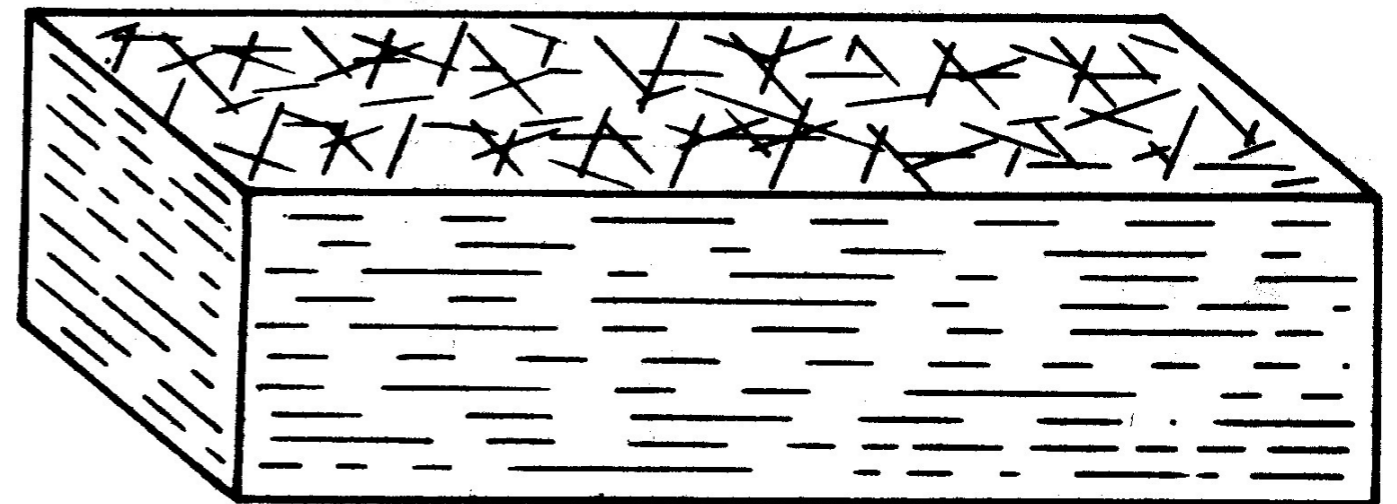
C fibers

C matrix

processed by laying down fibers in binder (pitch); high heat converts binder to C matrix

carbon-carbon composite

Randomly oriented fibers layered in 2D, not continuous with composite

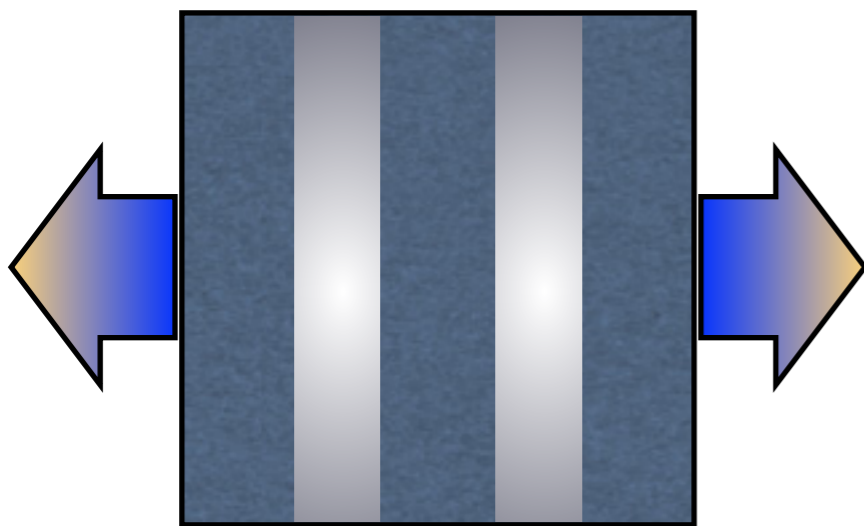


Stress / strain response depends on

- material properties (E , TS, σ_Y) of matrix + reinforcement
- amount of matrix + reinforcement (V_m , $V_r=1-V_m$)
- orientation of reinforcement relative to load
- size and distribution of reinforcement
- geometry (length of fibers, cross-sectional shape, aspect ratio)

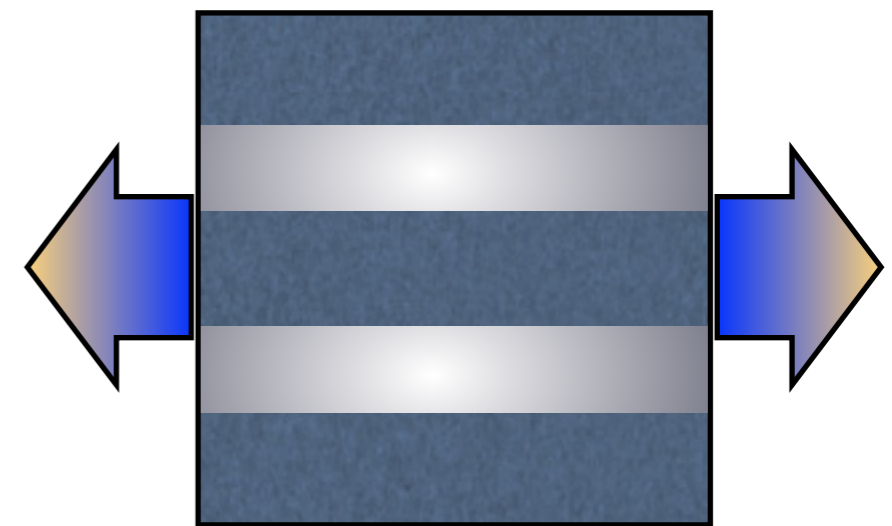
Two limiting cases for analysis:

isoload/isostress

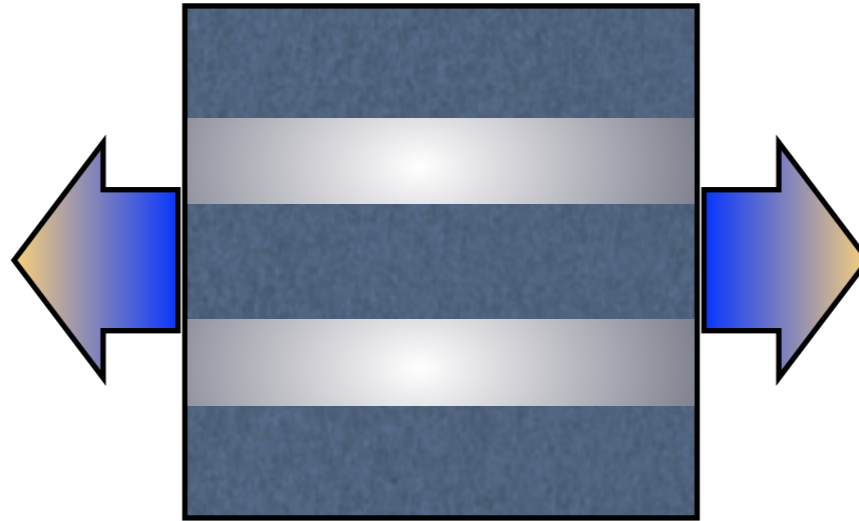


equal load in phases

isostrain



equal strain in phases



equal length/strain in phases

$$l_{\text{reinforcement}} = l_{\text{matrix}} = l_{\text{composite}}$$

$$\epsilon_{\text{reinforcement}} = \epsilon_{\text{matrix}} = \epsilon_{\text{composite}}$$

shared load:

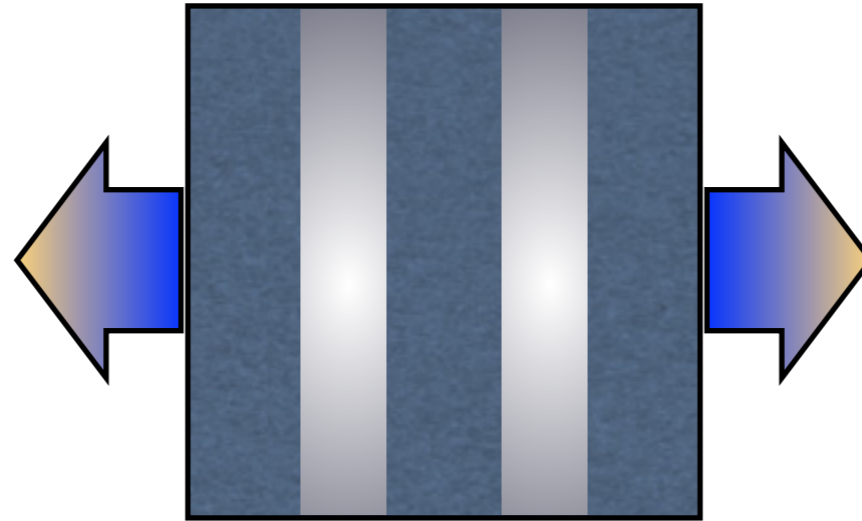
$$F_c = F_m + F_r$$

$$\frac{F_c}{A} = \frac{F_m}{A} + \frac{F_r}{A}$$

$$\sigma_c = \frac{F_m}{A_m} \frac{A_m}{A} + \frac{F_r}{A_r} \frac{A_r}{A}$$

$$\sigma_c = V_m \sigma_m + V_r \sigma_r \quad \text{ROM for stresses}$$

Similar to Voigt average



equal load/stress in phases

$$F_{\text{reinforcement}} = F_{\text{matrix}} = F_{\text{composite}}$$

$$\sigma_{\text{reinforcement}} = \sigma_{\text{matrix}} = \sigma_{\text{composite}}$$

shared length: $l'_c = l'_m + l'_r$

$$l_c(1 + \varepsilon_c) = l_m(1 + \varepsilon_m) + l_r(1 + \varepsilon_r)$$

$$1 + \varepsilon_c = V_m(1 + \varepsilon_m) + V_r(1 + \varepsilon_r)$$

$$\varepsilon_c = V_m \varepsilon_m + V_r \varepsilon_r \quad \text{ROM for strains}$$

Similar to Reuss average

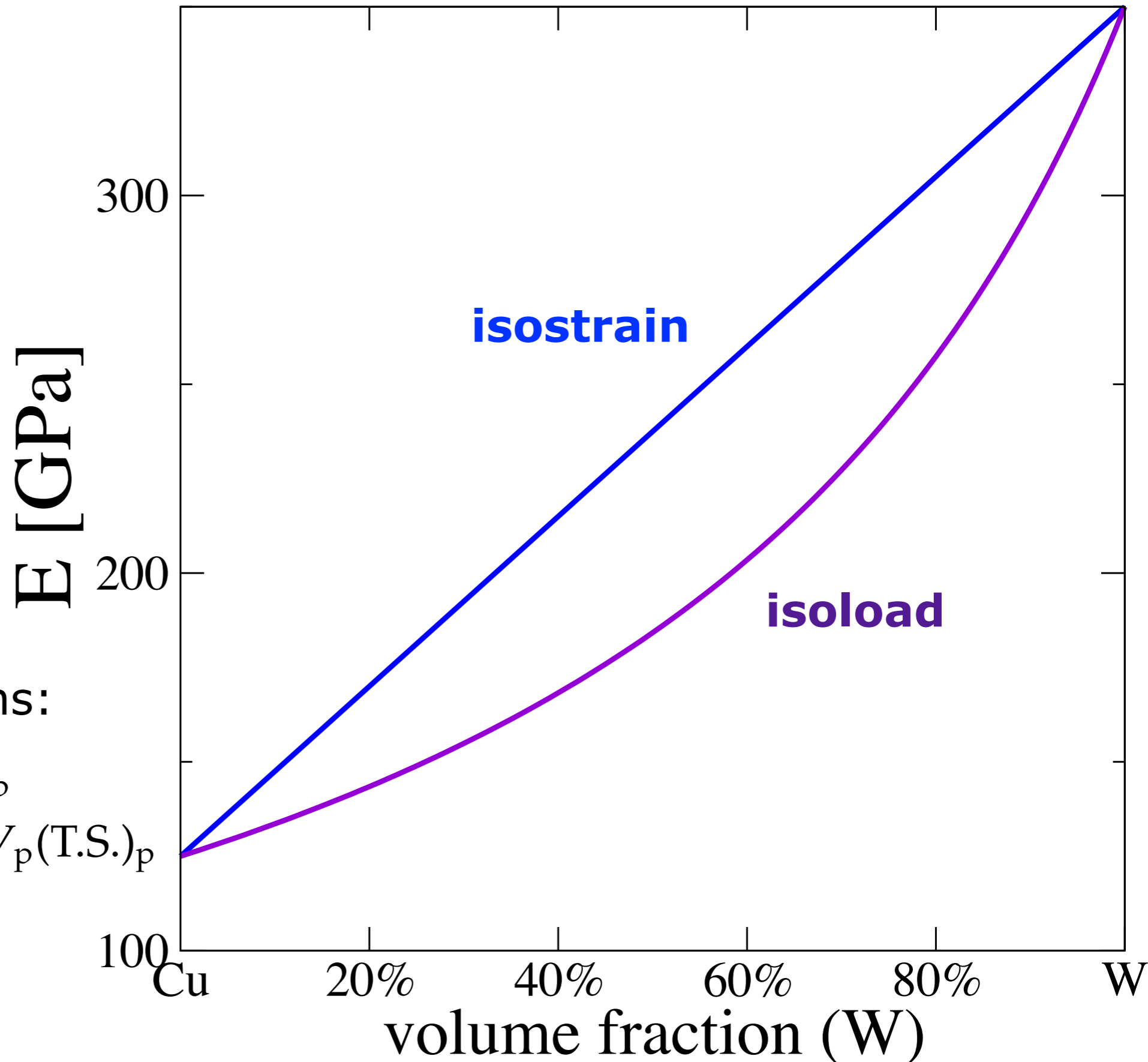
Elastic moduli of the composite are constrained by two limits:
isostrain and **isoload**

Empirical relations:

$$E_c = V_m E_m + K_c V_p E_p$$

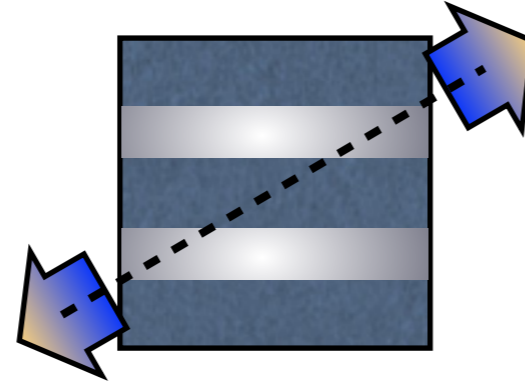
$$(T.S.)_c = V_m (T.S.)_m + K_s V_p (T.S.)_p$$

$$K_c \neq K_s < 1$$



Tensile stress not parallel to fibers has complex stress state:

3 limiting cases:



$$\underline{\sigma} = \begin{pmatrix} \sigma \cos^2 \theta & \sigma \cos \theta \sin \theta \\ \sigma \cos \theta \sin \theta & \sigma \sin^2 \theta \end{pmatrix}$$

1. small misorientation: limited by fiber failure ($\sigma_{\parallel} = \sigma \cos^2 \theta$)

$$(\text{T.S.})_c = \frac{\sigma_{\parallel}^{\star}}{\cos^2 \theta}$$

2. large misorientation: limited by matrix tensile failure ($\sigma_{\perp} = \sigma \sin^2 \theta$)

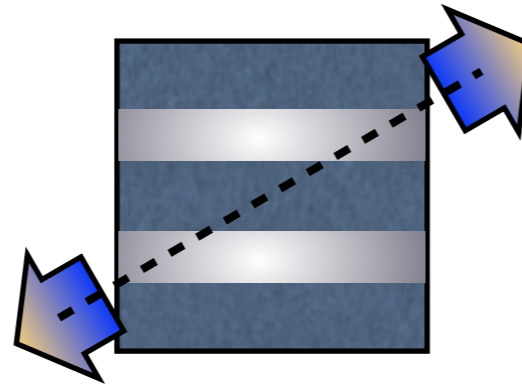
$$(\text{T.S.})_c = \frac{\sigma_{\perp}^{\star}}{\sin^2 \theta}$$

3. medium misorientation: limited by matrix shear failure ($\tau = \sigma \cos \theta \sin \theta$)

$$(\text{T.S.})_c = \frac{\tau_{m,y}}{\cos \theta \sin \theta}$$

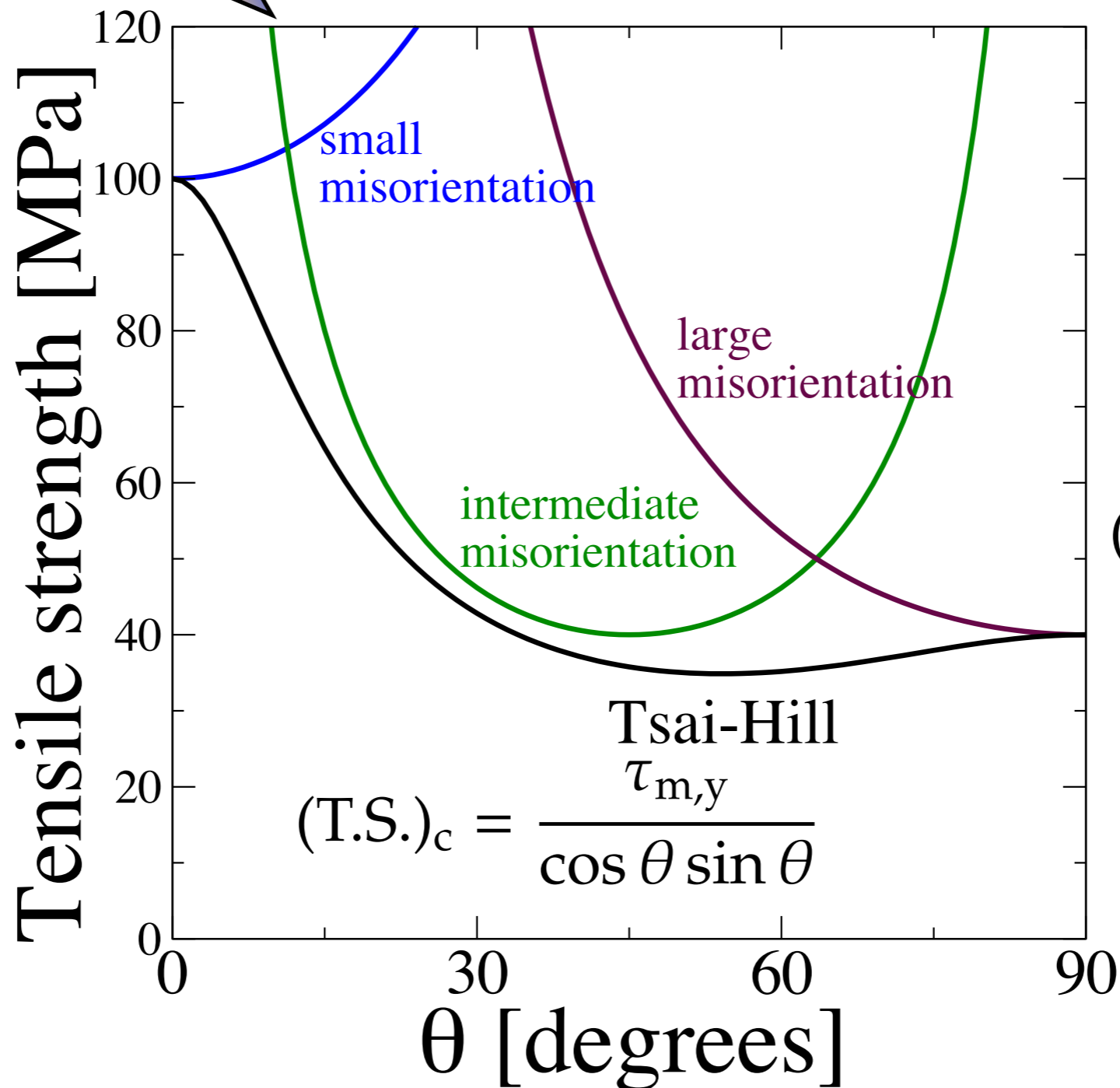
Tensile stress not parallel to fibers has complex stress state:

3 limiting cases:



$$\underline{\sigma} = \begin{pmatrix} \sigma \cos^2 \theta & \sigma \cos \theta \sin \theta \\ \sigma \cos \theta \sin \theta & \sigma \sin^2 \theta \end{pmatrix}$$

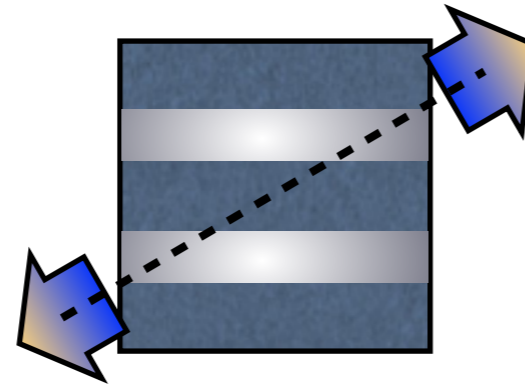
$$(\text{T.S.})_c = \frac{\sigma_{\parallel}^{\star}}{\cos^2 \theta}$$



$$(\text{T.S.})_c = \frac{\sigma_{\perp}^{\star}}{\sin^2 \theta}$$

Tensile stress not parallel to fibers has complex stress state:

Some limitations:



$$\underline{\sigma} = \begin{pmatrix} \sigma \cos^2 \theta & \sigma \cos \theta \sin \theta \\ \sigma \cos \theta \sin \theta & \sigma \sin^2 \theta \end{pmatrix}$$

1. Predicts that tensile strength *increases* for small misorientation.
2. Predicts “cusps” in strength vs. misorientation angle.
3. Doesn't account for multiaxial loading effects.

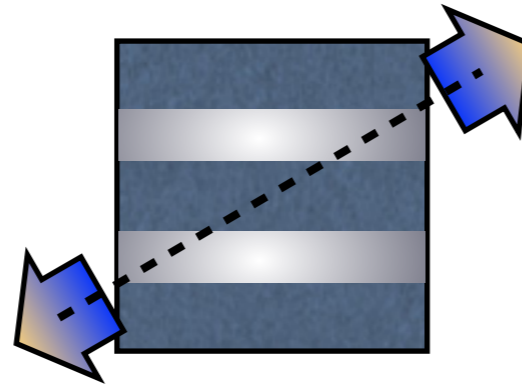
Solution: Tsai-Hill failure criterion:

$$\left(\frac{\sigma_{\parallel}}{\sigma_{\parallel}^*} \right)^2 - \left(\frac{\sigma_{\parallel} \sigma_{\perp}}{\sigma_{\perp}^{*2}} \right) + \left(\frac{\sigma_{\perp}}{\sigma_{\perp}^*} \right)^2 + \left(\frac{\tau}{\tau_{m,y}} \right)^2 = 1$$

$$(\text{T.S.})_c = \left[\frac{\cos^4 \theta}{\sigma_{\parallel}^{*2}} + \frac{\sin^4 \theta}{\sigma_{\perp}^{*2}} + \cos^2 \theta \sin^2 \theta \left(\frac{1}{\tau_{m,y}^2} - \frac{1}{\sigma_{\parallel}^{*2}} \right) \right]^{-1/2}$$

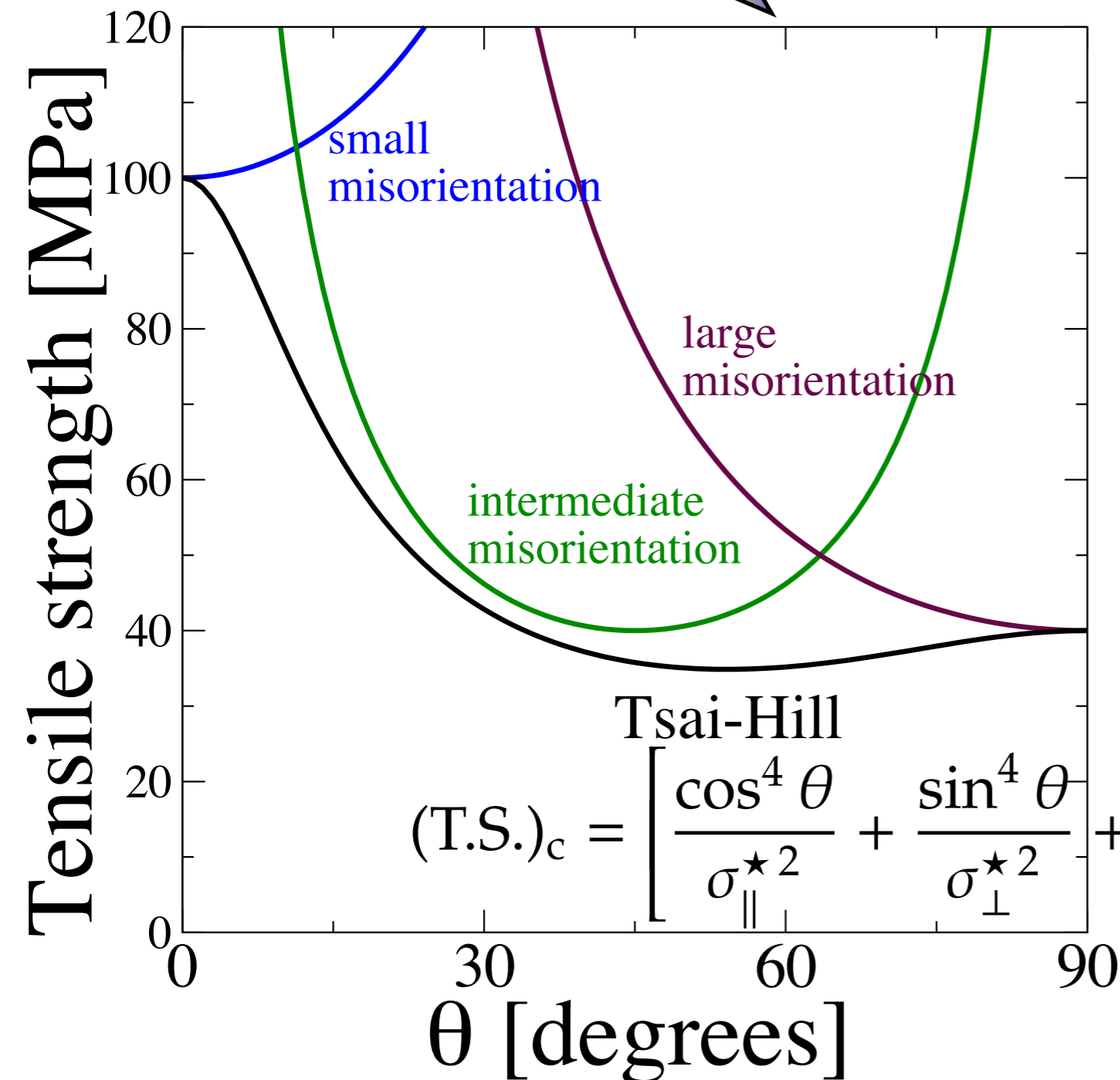
Orientation effects on tensile strength

Tensile stress not parallel to fibers has complex stress state:



$$\underline{\sigma} = \begin{pmatrix} \sigma \cos^2 \theta & \sigma \cos \theta \sin \theta \\ \sigma \cos \theta \sin \theta & \sigma \sin^2 \theta \end{pmatrix}$$

Tsai-Hill smooths out cusps
Never exceeds aligned T.S.



$$(T.S.)_c = \left[\frac{\cos^4 \theta}{\sigma_{\parallel}^{*2}} + \frac{\sin^4 \theta}{\sigma_{\perp}^{*2}} + \cos^2 \theta \sin^2 \theta \left(\frac{1}{\tau_{m,y}^2} - \frac{1}{\sigma_{\parallel}^{*2}} \right) \right]^{-1/2}$$