

Problem 1. Consider a linear, time-invariant system with the following impulse response:

$$\dot{y} = -5y + 2r$$

where r is the reference signal.

- (5 points) Compute the transfer function from R to Y . What are the poles, zeros, and DC gain?
- (3 points) Draw a block diagram for this system using integrator, summation, and gain blocks.
- (2 points) What is the general form of the free response?
- (2 points) What is the general form of the forced response?
- (4 points) Assuming $y(0) = 0$. Compute the step response by hand.
- (4 points) Now suppose that the reference signal is the ramp: $r(t) = t$ for $t \geq 0$. Assuming $y(0) = 0$. Compute the response $y(t)$ by hand.

Solution.

- From the ODE, we have $sY(s) + 5Y(s) = 2R(s)$, therefore, the transfer function from R to Y is:

$$H(s) = \frac{Y(s)}{R(s)} = \frac{2}{s + 5}.$$

The pole of the above transfer function is $s = -5$, there is no zero, and the DC gain is $H(0) = \frac{2}{5}$.

- The block diagram for this system can be found in Figure 1.
- Since we have one pole $s = -5$, the general form of the free response is: $y(t) = Ce^{-5t}$, where C can be determined by the initial condition $y(0)$.
- The general form of the forced response is $y(t) = y_p(t) + Ce^{-5t}$, where $y_p(t)$ is a particular solution of the above ODE. C can be determined by the initial condition $y(0)$.
- Assume that $y_p(t) = C_0$, which is a constant. Then we have $\dot{y}_p(t) = 0$. Substituting this into the ODE leads to $0 = -5C_0 + 2$, which gives $C_0 = \frac{2}{5}$. Therefore, we have $y(t) = \frac{2}{5} + Ce^{-5t}$. Since $y(0) = 0$,

$$\frac{2}{5} + Ce^0 = \frac{2}{5} + C = 0 \implies C = -\frac{2}{5}.$$

Hence $y(t) = \frac{2}{5} - \frac{2}{5}e^{-5t}$.

- Now assume that $y_p(t) = k_0 + k_1t$, $\dot{y}_p(t) = k_1$. Substituting this into the ODE gives

$$k_1 = -5k_0 - 5k_1t + 2t \implies \begin{cases} k_0 = -\frac{2}{25} \\ k_1 = \frac{2}{5} \end{cases}$$

Hence we have $y(t) = -\frac{2}{25} + \frac{2}{5}t + Ce^{-5t}$. Since $y(0) = 0$:

$$-\frac{2}{25} + Ce^0 = 0 \implies C = \frac{2}{25}.$$

Hence $y(t) = -\frac{2}{25} + \frac{2}{5}t + \frac{2}{25}e^{-5t}$.

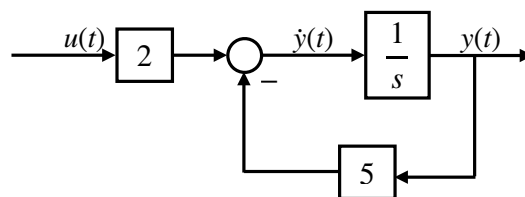


Figure 1: Block diagram of the system

Problem 2.

- (a) (5 points) Consider the following second-order system.

$$H(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

For the step response, suppose we have the following specifications:

- rise time $t_r \leq 1.8$
- settling time $t_s \leq 6.5$

Plot the pole locations that achieve this on the axes provided below. **Clearly label your axes.** What conditions must be satisfied by the ω_n and ζ parameters to meet these specifications? *Be sure to state your conditions in terms of ω_n and ζ !*

- (b) (5 points) Consider the following second-order system.

$$H(s) = \frac{3}{s^2 + 4s + 3}$$

Does the step response of this system satisfy the following specifications?

- rise time $t_r \leq 1.8$
- settling time $t_s \leq 6.5$

- (c) For the second-order system in (b). Now consider

$$y(0) = 1 \quad \dot{y}(0) = 2$$

$$u(t) = \begin{cases} 2 - 2e^{-t} & t \geq 0 \\ 0 & t < 0 \end{cases}$$

Apply Final Value Theorem (FVT) to calculate the steady state value of y . Provide detailed explanations justifying how FVT is applied in this case. (You are allowed to verify your answers using `Matlab` but you need to first provide all the derivations using FVT.)

Solution.

- (a) From the time specs defined in the question, we have:

$$t_r \leq 1.8 \implies \frac{1.8}{\omega_n} \leq 1.8 \implies \omega_n \geq 1$$

$$t_s \leq 6.5 \implies \frac{3}{\zeta\omega_n} \leq 6.5 \implies \zeta\omega_n \geq \frac{6}{13}$$

The poles of the above second order system can be written as: $s_{1,2} = -\zeta\omega_n \pm j\omega_n\sqrt{1 - \zeta^2}$. To ensure the stability, we need $\zeta\omega_n > 0$. To ensure the above time specs, we need the real part of the poles $\text{Re}\{s_{1,2}\} = -\zeta\omega_n \leq -\frac{6}{13}$. The magnitude of the poles $|s_{1,2}| = \omega_n \geq 1$. The shaded area of Figure 2 shows the admissible pole locations.

- (b) For this specific system, we have $\omega_n = \sqrt{3} > 1$ and $-\zeta\omega_n = -2 < \frac{6}{13}$. Therefore, this system satisfies the required specifications.
- (c) The system can be written as the following ODE form:

$$\ddot{y}(t) + 4\dot{y}(t) + 3y(t) = 3u(t).$$

Noticing that in this question we have non-zero initial conditions, performing the Laplace transform of the above ODE gives:

$$s^2Y(s) - s - 2 + 4sY(s) - 4 + 3Y(s) = 3U(s) \implies Y(s) = \frac{3U(s) + s + 6}{s^2 + 4s + 3}.$$

Since we have $u(t) = 2 - 2e^{-6}, t \geq 0$, $U(s) = \frac{2}{s} - \frac{2}{s+1} = \frac{2}{s(s+1)}$. Therefore, we have

$$sY(s) = s \frac{3U(s) + s + 6}{s^2 + 4s + 3} = \frac{6 + s(s+1)(s+6)}{(s+1)^2(s+3)},$$

which has all strictly stable poles, hence FVT can be applied. Specifically, we have:

$$\lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} sY(s) = \lim_{s \rightarrow 0} \frac{6 + s(s+1)(s+6)}{(s+1)^2(s+3)} = 2.$$

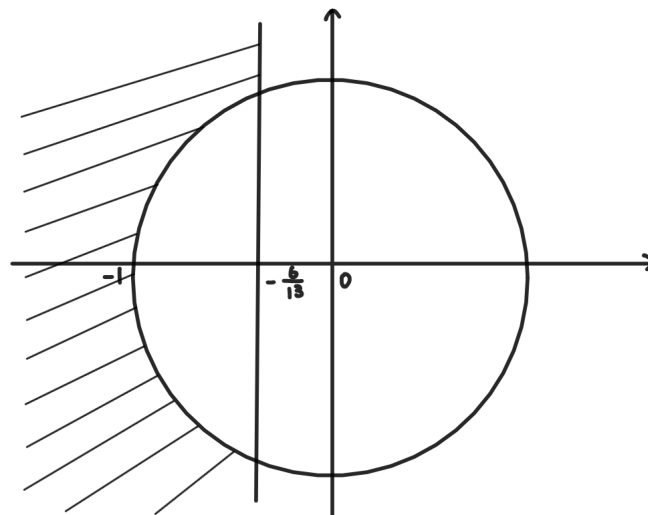


Figure 2: Admissible pole location for 2(b).

Problem 3.

- (a) (10 points) Use the Routh-Hurwitz criterion to evaluate the stability of a system with the following transfer function:

$$T(s) = \frac{10}{s^5 + 3s^4 + 3s^3 + 7s^2 + 2s + 1}$$

Be sure to give the Routh-Hurwitz table as well as the number of unstable poles.

- (b) (10 points) Now consider a different transfer function:

$$T(s) = \frac{10}{s^5 + 3s^4 + 5s^3 + 7s^2 + 2s + 1}$$

Use the Routh-Hurwitz criterion to evaluate the stability of the above system. Again, do the calculations by hand.

Solution.

(a) We have the following:

s^5	s^4	s^3	s^2	s^1	s^0
1	3	2/3	-1/2	3	1
3	7	5/3	1	0	0
2	1	0	0	0	0

Table 1: Routh table for part (a) - note the table is transposed

Since there is 2 sign changes in the first row, there are two unstable poles.

(b) We have the following:

s^5	s^4	s^3	s^2	s^1	s^0
1	3	8/3	41/8	47/41	1
5	7	5/3	1	0	0
2	1	0	0	0	0

Table 2: Routh table for part (b) - note the table is transposed

Since there are no sign changes in the first row, this system is stable.

Problem 4. Let $G(s) = \frac{15}{s^2 - 5s + 7}$. The output of G is denoted as y , and the reference is denoted as r . Suppose $e = r - y$. And the input to G is denoted as u .

- (5 points) What is the closed-loop transfer function from r to y if we use the PI controller $u(t) = K_p e(t) + K_i \int_0^t e(\tau) d\tau$? Is it possible to choose (K_p, K_i) such that the closed-loop transfer function from r to y is stable? Justify your answers using Routh-Hurwitz criterion.
- (5 points) What is the closed-loop transfer function from r to y if we use the PID controller $u(t) = K_p e(t) + K_i \int_0^t e(\tau) d\tau + K_d \dot{e}(t)$?
- (5 points) How to choose the PID control gains (from (b)) such that the closed-loop transfer function from r to y has repeated poles at -4 ? (In other words, all the poles of the closed-loop transfer function from r to y are equal to -4 .)
- (5 points) How to choose the PID control gains (from (b)) such that the closed-loop transfer function from r to y has repeated poles at -2 ? (In other words, all the poles of the closed-loop transfer function from r to y are equal to -2 .)
- (5 points) Compare the controllers designed in (c) and (d). Which one has a faster unit step response (assuming zero initial conditions)? Which one can reject the sensor noise in a better way?

Solution.

- (a) Given above $G(s)$ for a PI controller we have that $U(s) = K_p + K_i/s$ and the closed loop transfer function is given by $H(s) = \frac{U(s)G(s)}{1 + U(s)G(s)}$. Expanded we get:

$$H(s) = \frac{15(K_i + K_p s)}{15K_i + (15K_p + 7)s - 5s^2 + s^3}$$

We have no control over the s^2 term whose coefficient violates the necessary condition for stability. Therefore it is not possible to choose K_p, K_i to achieve stability.

(b) Now we have that $U(s) = K_p + K_i/s + K_d s$ and the closed loop transfer function is:

$$H(s) = \frac{15(K_i + K_p s + K_d s^2)}{15K_i + (7 + 15K_p)s + (15K_d - 5)s^2 + s^3}$$

(c) For all poles at -4 we need the following characteristic equation:

$$(s + 4)^3 = s^3 + 12s^2 + 48s + 64$$

to be the same as the denominator of $H(s)$ above. Then comparing terms we get

$$\begin{aligned} 15K_i &= 64 \implies K_i = \frac{64}{15} \\ 7 + 15K_p &= 48 \implies K_p = \frac{41}{15} \\ 15K_d - 5 &= 12 \implies K_d = \frac{17}{15} \end{aligned}$$

(d) By a similar procedure for all poles at -2 , we need characteristic equation $s^3 + 6s^2 + 12s + 8$. Therefore,

$$K_i = \frac{8}{15} \quad K_p = \frac{1}{3} \quad K_d = \frac{11}{15}$$

(e) The poles of the controller (c) are more negative than those of controller (d) and so the former will have a faster step response, see Figure 3. However, this comes at the cost of larger K_p, K_i, K_d gains (as seen above) which may also amplify noise. Therefore controller (d) will reject sensor noise better.

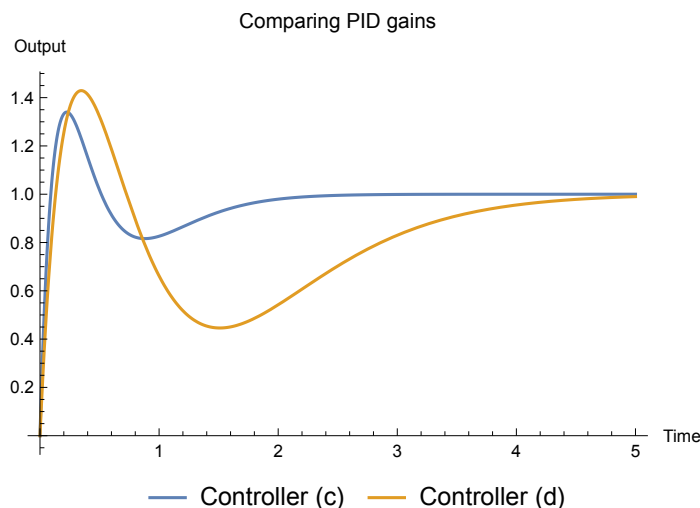
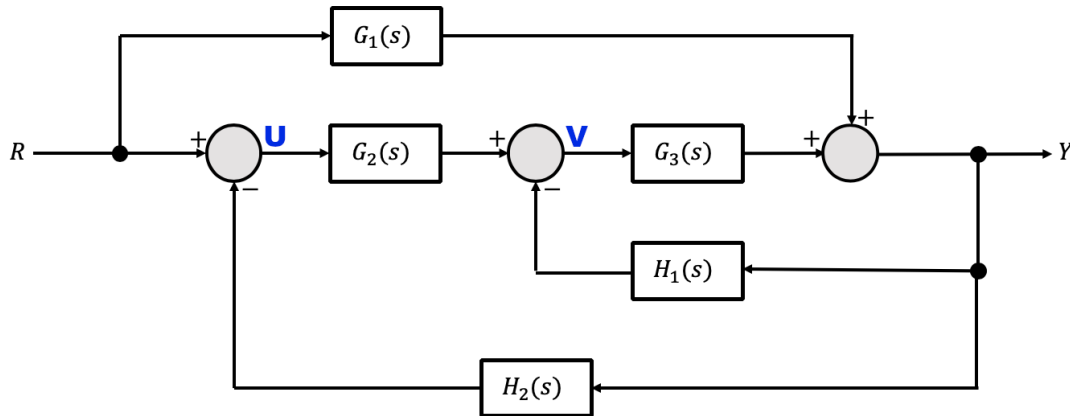


Figure 3: Comparing the step response of controllers,

Problem 5.

- (a) (5 points) Determine the transfer function $\frac{Y(s)}{R(s)}$ from the block diagram below. (All G_i and H_i are single-input, single-output systems.)
- (b) (5 points) Consider the following system:

$$G(s) = \frac{15808}{s^5 + 50s^4 + 837s^3 + 5040s^2 + 10336s + 16640}$$



Construct a first-order or second-order approximation from the dominant pole. Do you expect the dominant pole approximation to be accurate? Use Matlab (or another numerical tool) to plot the step responses for $G(s)$ and your dominant pole approximation on a single figure.

Solution.

- (a) Define the quantities denoted in the figure as follows:

$$\begin{aligned} U(s) &= R(s) - H_2(s)Y(s) \\ V(s) &= G_2(s)U(s) - H_1(s)Y(s) \\ Y(s) &= G_3(s)V(s) + G_1(s)R(s) \end{aligned}$$

Then

$$\begin{aligned} Y(s) &= G_3(s) (G_2(s)U(s) - H_1(s)Y(s)) + G_1(s)R(s) \\ &= G_3(s) (G_2(s) (R(s) - H_2(s)Y(s)) - H_1(s)Y(s)) + G_1(s)R(s) \end{aligned}$$

Therefore,

$$\frac{Y(s)}{R(s)} = \frac{G_3(s)G_2(s) + G_1(s)}{1 + G_3(s)G_2(s)H_2(s) + G_3(s)H_1(s)}$$

- (b) Factor the denominator as:

$$s^5 + 50s^4 + 837s^3 + 5040s^2 + 10336s + 16640 = (s + 8) (s^2 + 2s + 5) (s^2 + 40s + 416)$$

yielding that the dominant poles $\{-1 \pm 2i\}$ belong to the second factor $(s^2 + 2s + 5)$. Then construct a second order approximation as

$$\hat{G}(s) = \frac{z_0}{(s^2 + 2s + 5)}$$

with $z_0 = \frac{19}{4}$ obtained by setting $\hat{G}(0) = G(0)$.

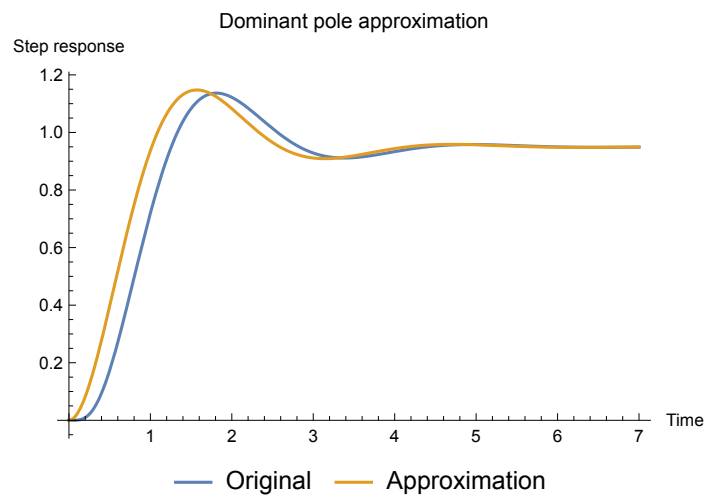


Figure 4: Step response of the system and its approximation